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Residential Zoned Systems

It's Different Now

PG&E's Energy Training Center-Stockton



***Pacific Gas and
Electric Company***[®]

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Introduction



This Class

- Residential Zoning Practices
- Zoning Within Title 24
- Duct Design Basics
- Current Research and Zoning Methods
- Ducted Zoning
- Non-Ducted Zoning

Questions

- What Air Conditioners do you work on?
- How do you know if an Air Conditioner is working efficiently?
- What are the two things that an Air Conditioner removes from inside air?
- What are the climate differences between California and Florida?
- How does the outside air affect the inside air?

My Answers

- What Air Conditioners do you work on?
 - Residential and Commercial, Split, Package, Window, Mini-Split, Domestic and Foreign
- How do you know if an Air Conditioner is working efficiently?
 - Test the airflow, T' Split, Watts, Superheat, Subcooling, and compare to conditions
- What are the two things that an Air Conditioner removes from inside air?
 - Sensible Heat (temperature) and Humidity
- What are the climate differences between California and Florida?
 - Humidity
- How does the outside air affect the inside air?

Questions

- What do you want a California Air Conditioner to do more of sensible cooling or dehumidification?
- What are the ways to make it do that?
- What make it worse?

My Answers

- What do you want a California Air Conditioner to do more of sensible cooling or dehumidification?
 - Sensible Cooling
- What are the ways to make it do that?
 - Increase Airflow
 - Evaporate the Water Off the Coil After the Compressor is Off
- What makes it worse?
 - Low Airflow
 - Bypasses
 - Fan Off at Compressor Off

This Class

The primary purpose of zoning air conditioners, heat pumps, and furnaces is to improve comfort.

Increased comfort is attained by having the capacity of the HVAC system (cooling or heating delivered) **follow the shift in load** as it changes across the house.

Should We Zone This?

Main House 1200 sq. Ft + Converted Detached Garage



Should We Zone This?



Why Zone a Residence?









Why Zone a Residence?

- A residence may be a candidate for zoning if cooling, heating, or dehumidification loads shift over time due to:
 - Solar Heat Gain
 - Occupancy
 - Use of the Area
 - Large Intermittent Internal Gains
 - Large Moisture Input
 - Tall Buildings
 - Above Grade and Below Grade
 - Summer vs. Winter

Once Upon A Time

The Old Days



Zoning

The primary purpose of zoning air conditioners, heat pumps, and furnaces is to improve comfort.

Increased comfort is attained by having the capacity of the HVAC system (cooling or heating delivered) **follow the shift in load** as it changes across the house.

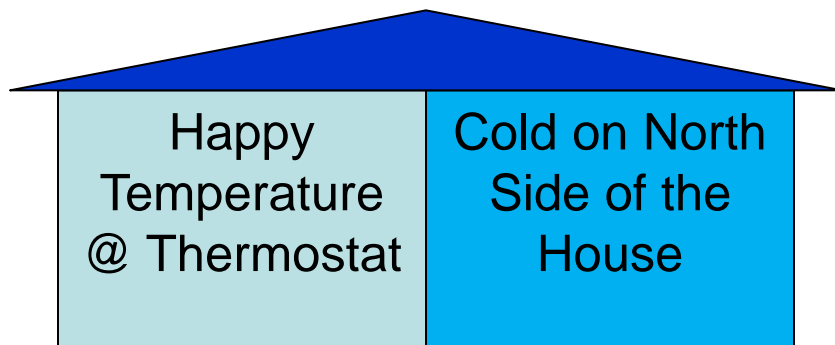
Is There Such a Thing as
Excess Air?

Is There Such a Thing as
Excess Capacity?

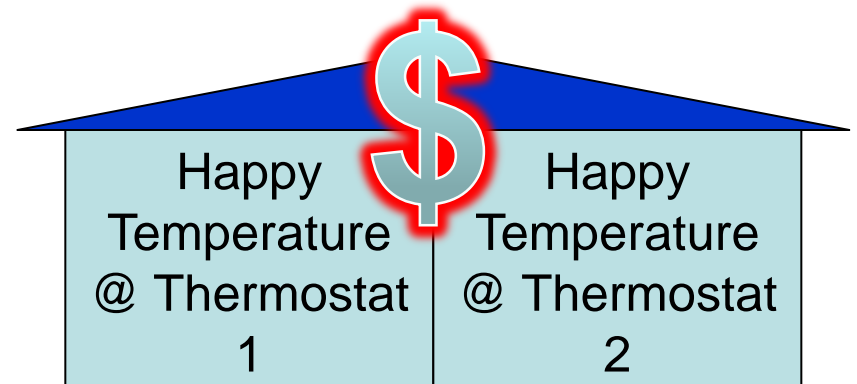
Is There Such a Thing as
Excess Efficiency?

Zone for Comfort

- Zoning can improve comfort for a structure that has significant variations in load
- Ducted Zoning is very unlikely to provide any reduction in energy bills and is capable of increasing the energy bills



Single Zone



Two Zones

An Example

- **NAHB Test House** (Kenney & Barbour 1994)
 - “Studies have demonstrated that a multi-zone system will use more energy than a central thermostat system when a constant setpoint is used.”
 - “A 35 percent increase was documented (Oppenheim 1991) as a direct result of a multi-zone system being more responsive to the cooling needs of the entire house..”

Zoning Criteria

- All portions of the zone should have a similar load pattern. A somewhat isolated West side for example.
- All portions of the zone should have a similar occupancy pattern. Sleeping areas vs. Daytime use for example.

Zoning Methods

- Heating Only Convectors
- Ductless Mini-Split Systems
- Multiple Systems
- Ducted Multi Capacity System
- Ductless Multi-Split Systems
- Ducted Mini- or Multi-Split Systems
- Ductless or Ducted Single Package (PTAC)
- **Ducted Single Capacity System**

Heating Only Convectors



Hydronic or Electric

Energy Savings is Possible with These

Ductless Mini-Split

- Single Speed
- Variable Speed
- One or more Heads
- Single Thermostat



Energy Savings is possible with these

Multiple Systems

- Relatively Simple Design
- Design the Duct System



Multi-Split Systems

- Two or More “Heads”
- Each Serving a Compatible Zone
- Two or More Thermostats
- Can Produce Lower Efficiency



Ducted Mini or Multi-Split

- DANGER – Static Pressure !!!
- Design the Duct System



PTAC & PTHP

- Small sleeve wall units have low efficiency, but higher efficiency than electric baseboards.
- Highest EERs are 12 to 13 (55 listed HP)
- 6000 BTU heat
7000 BTU cool



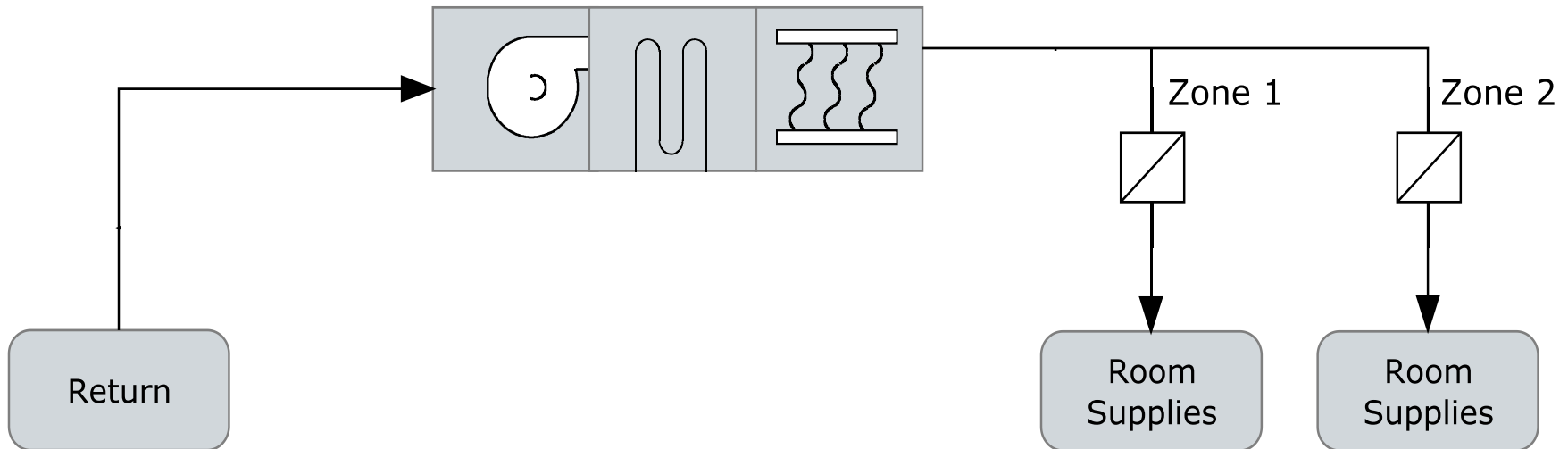
Ducted Systems First Priority

Reduce or Eliminate Differences

- Shading
- Low E Glass
- Infiltration Control (air sealing)
- Insulation
- Reduce the size of the Air Conditioner
- Work toward equal CFM in heating and cooling
- Design a Proper Duct System (reduce room to room temperature differences)

Multiple Capacity System

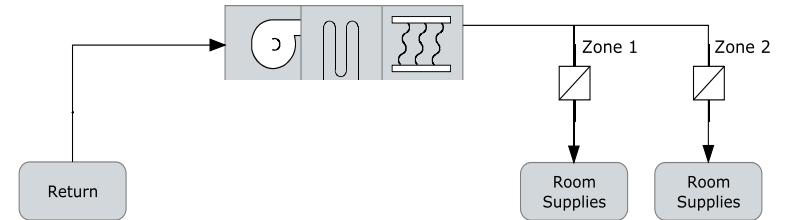
- Match Zones to AC Capacity at All Speeds
- Variable Speed Compressor and Fan



At least 350 CFM per Delivered Ton in all modes
Vary Compressor and Fan speeds with demand from zones
Limited by the range of the variable capacity

Multiple Capacity System

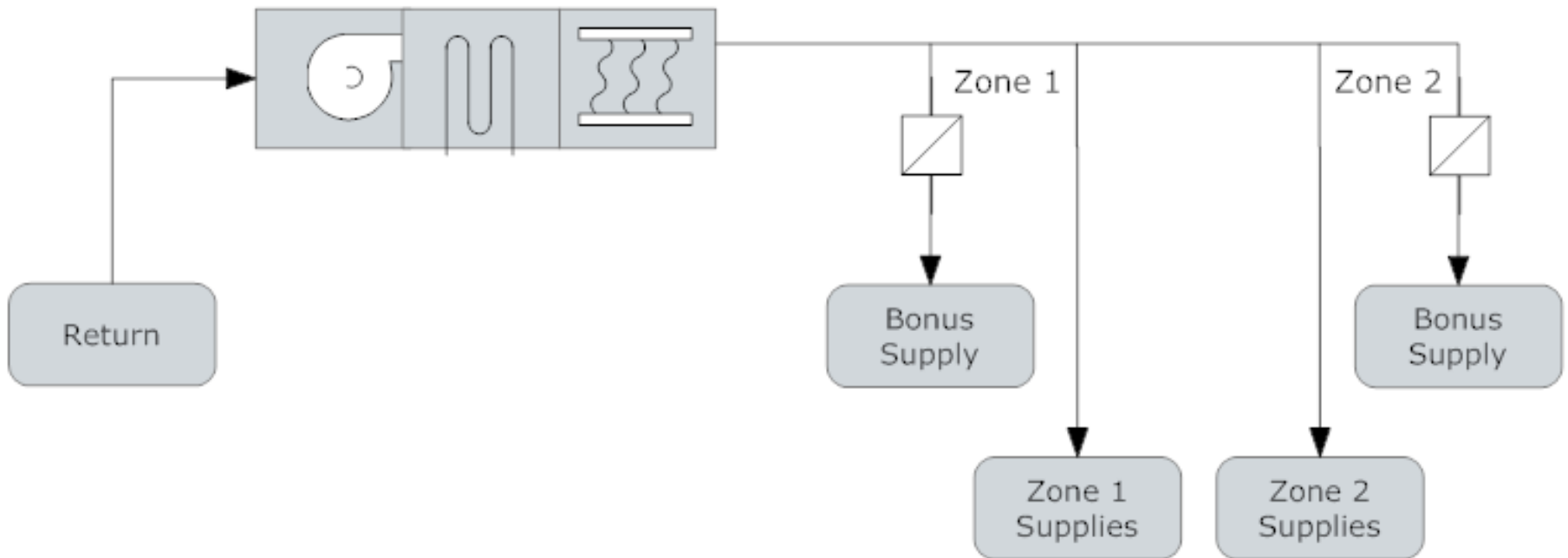
(2 or more compressor speeds) in Title 24



- At least 350 CFM per Nominal Ton **INTO THE RETURN** on Highest Compressor Speed with all Zones Calling
- Fan Watt Draw $\leq .58$ W per CFM under above conditions
- Additional performance credit available if higher airflow and/or better fan efficacy is modeled and verified under the above conditions
- A bypass may be used if it is modeled in the compliance software

A Good Single Capacity System

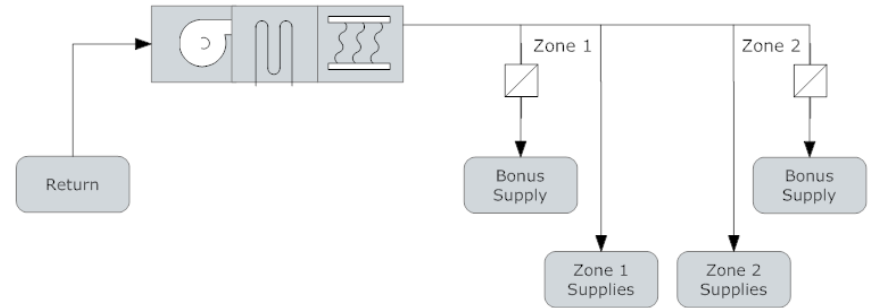
- Move Part of the Capacity from One Zone to Another based on Load



At least 350 CFM per Ton in All modes

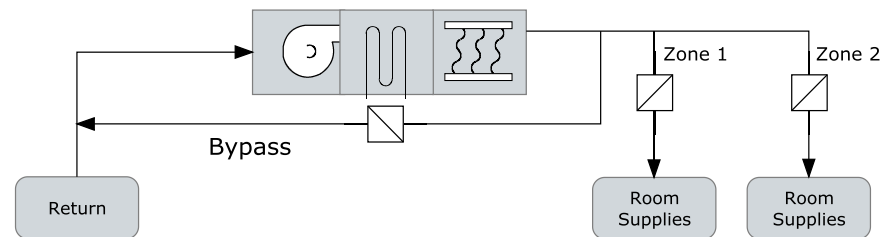
A Single Capacity System in Title 24

- A. Prescriptive Compliance: At least 350 CFM per Nominal Ton **INTO THE RETURN** in all modes. Bypass Duct Not Allowed



- B. Performance Compliance: At least 350 CFM per Nominal Ton **INTO THE RETURN** WITH ALL ZONES CALLING.

This method carries with it a large efficiency penalty (assumes 150 CFM per ton). Bypass Duct Allowed



- C. Performance Compliance with Reduced Penalty or Additional Performance Credit: B above except more than 150 CFM per nominal ton and/or less than 0.58 W/CFM modeled and verified in all modes

Title 24 Questions

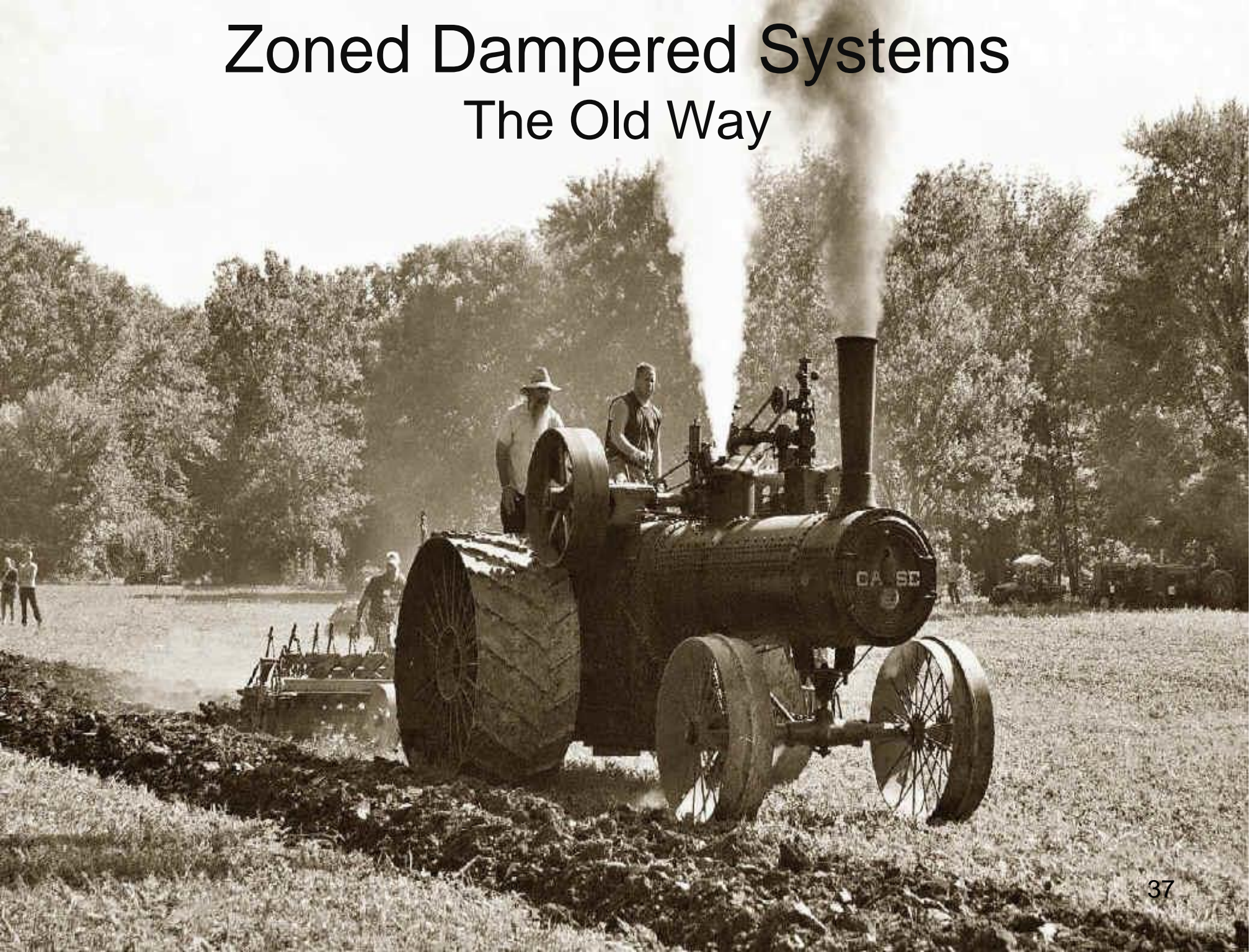
- “There used to be compliance credit requirements that included
 - a return in every zone and
 - no flow out of registers in zones not calling and
 - no more than 40 ft² non-closable opening area between zones.”

Question: “Do we have to meet those (on top of every thing else already discussed) when there is a zoned system?”

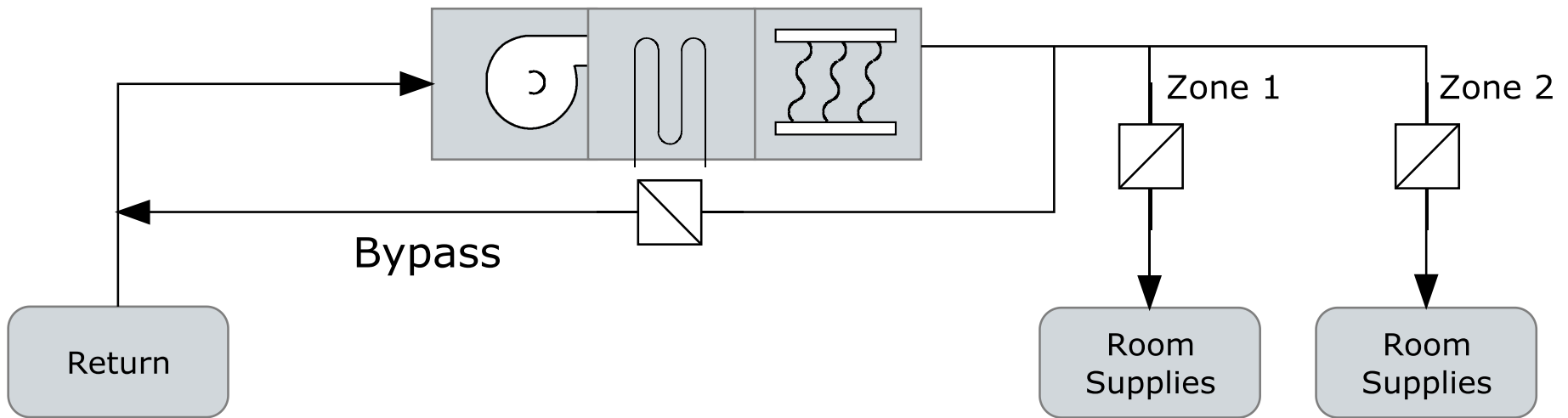
Answer: No the above requirements are only for heating where you are asking for a Title 24 credit for zoning in heating. If you are not asking for the credit they do not apply.

Zoned Dampened Systems

The Old Way



A Typical Dampered Multi-Zone AC System With Bypass Duct



What Does Research Show Us?

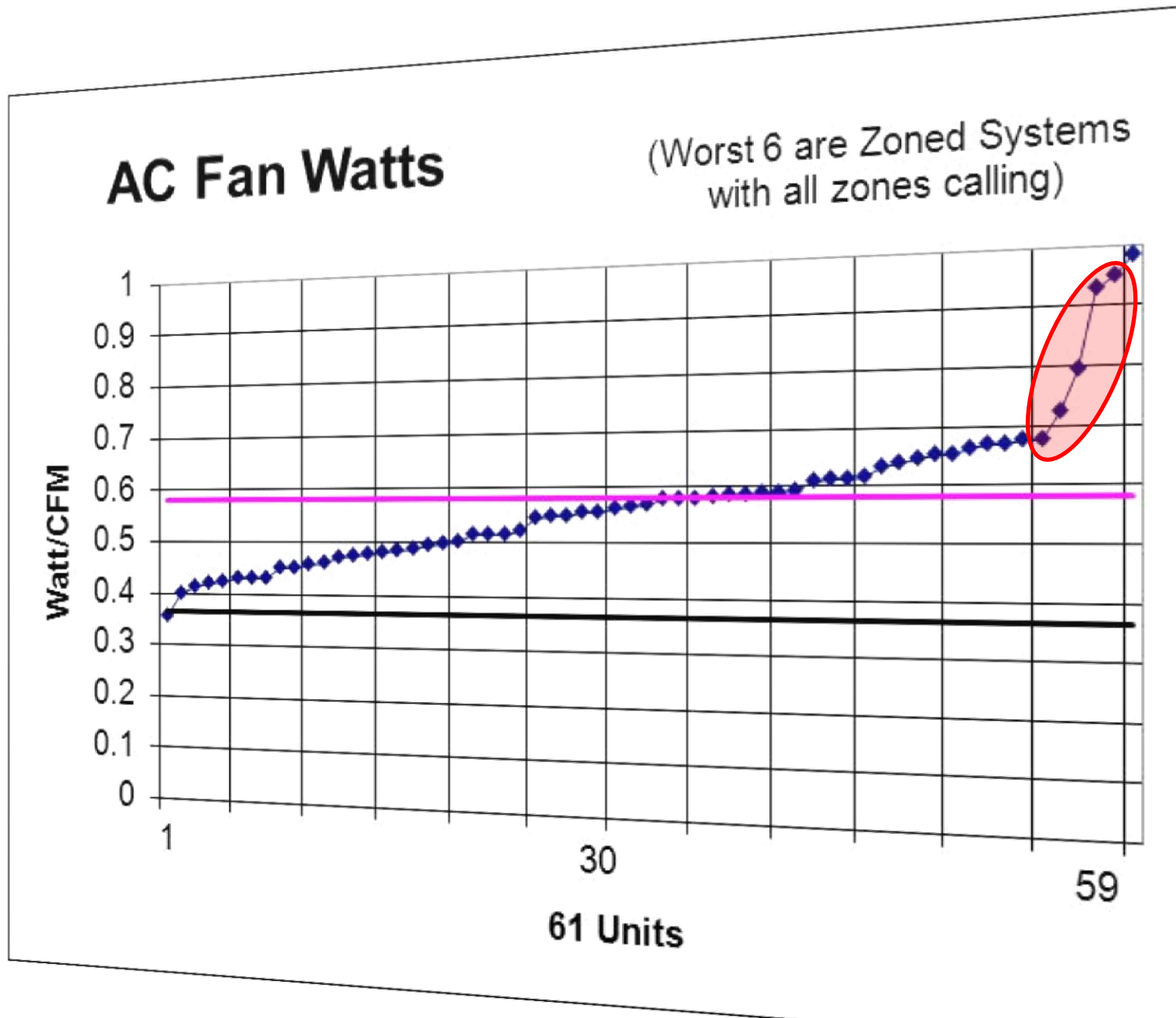


Efficiency Characteristics and Opportunities for New California Homes (ECO)

- Field survey of an 80 unit sample of new CA homes and found:
 - AC systems have low capacity and efficiency
 - Cooling duct pressures are very high
 - Cooling Fan Watts are high

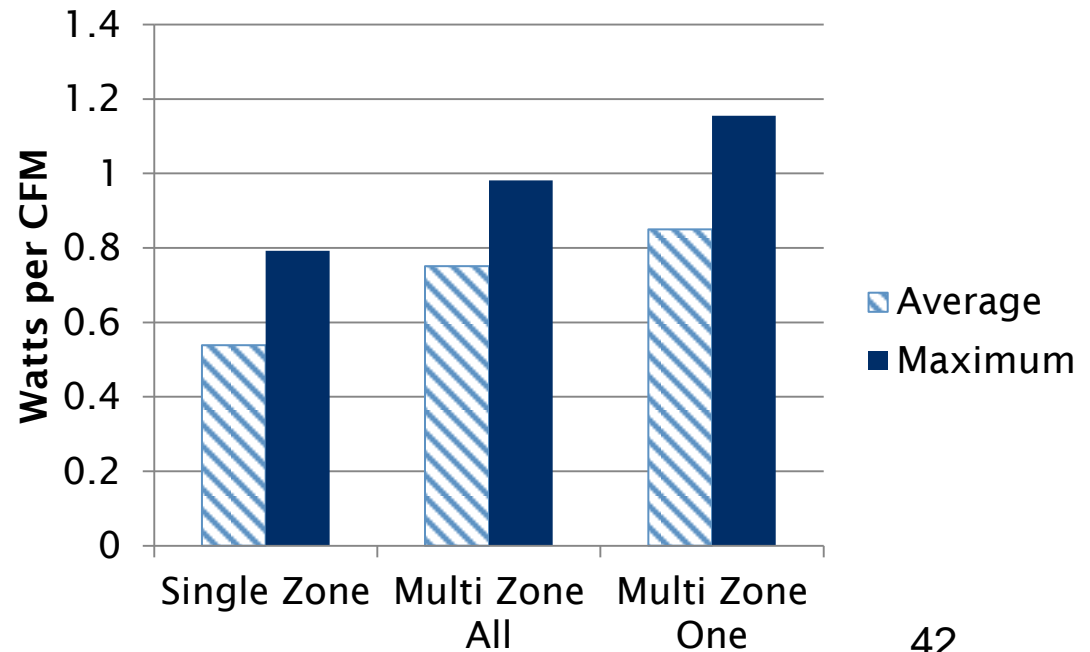
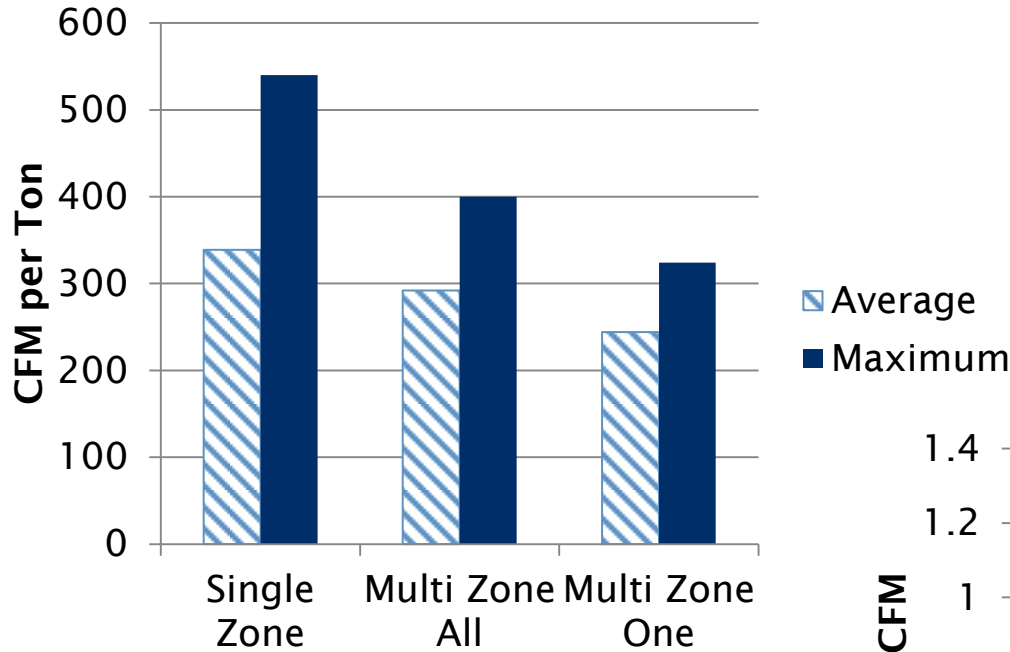
http://www.proctoreng.com/dnld/ECO_Report_CEC.pdf

Watts per Delivered CFM

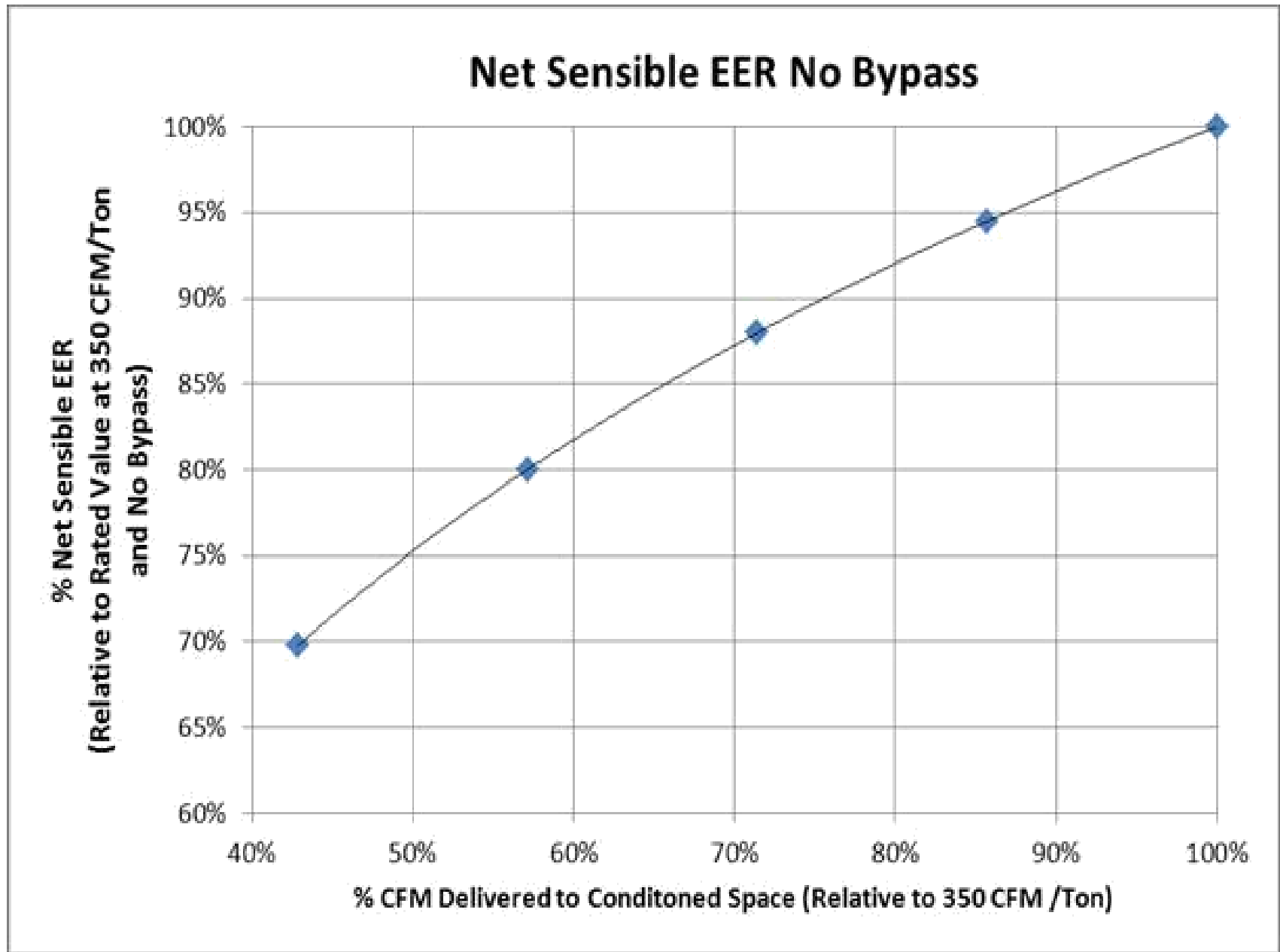


Dampened Multi-Zone Systems have Low CFM per Ton and High Cooling Fan Power per CFM

and Low Sensible
EER

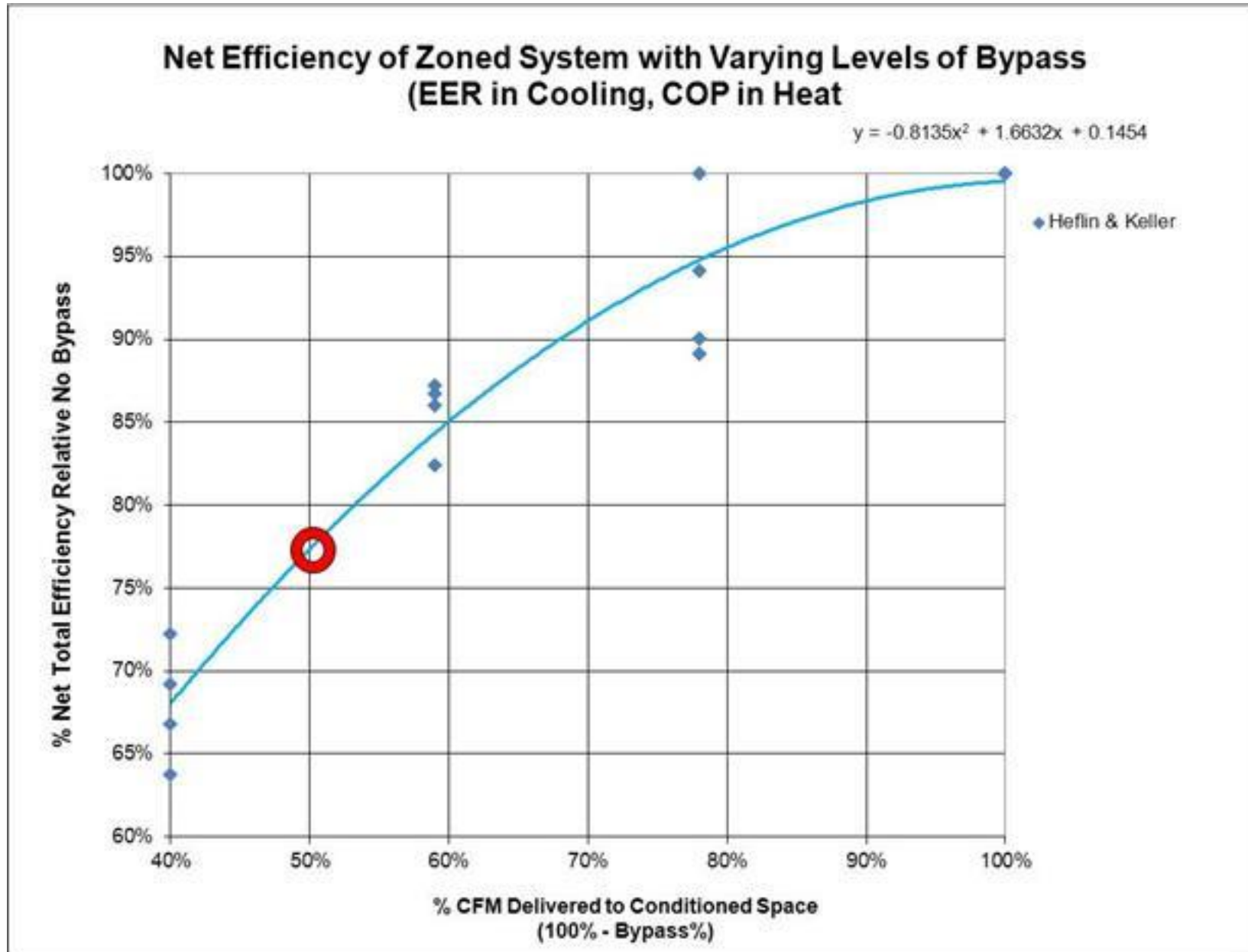


Low Airflow



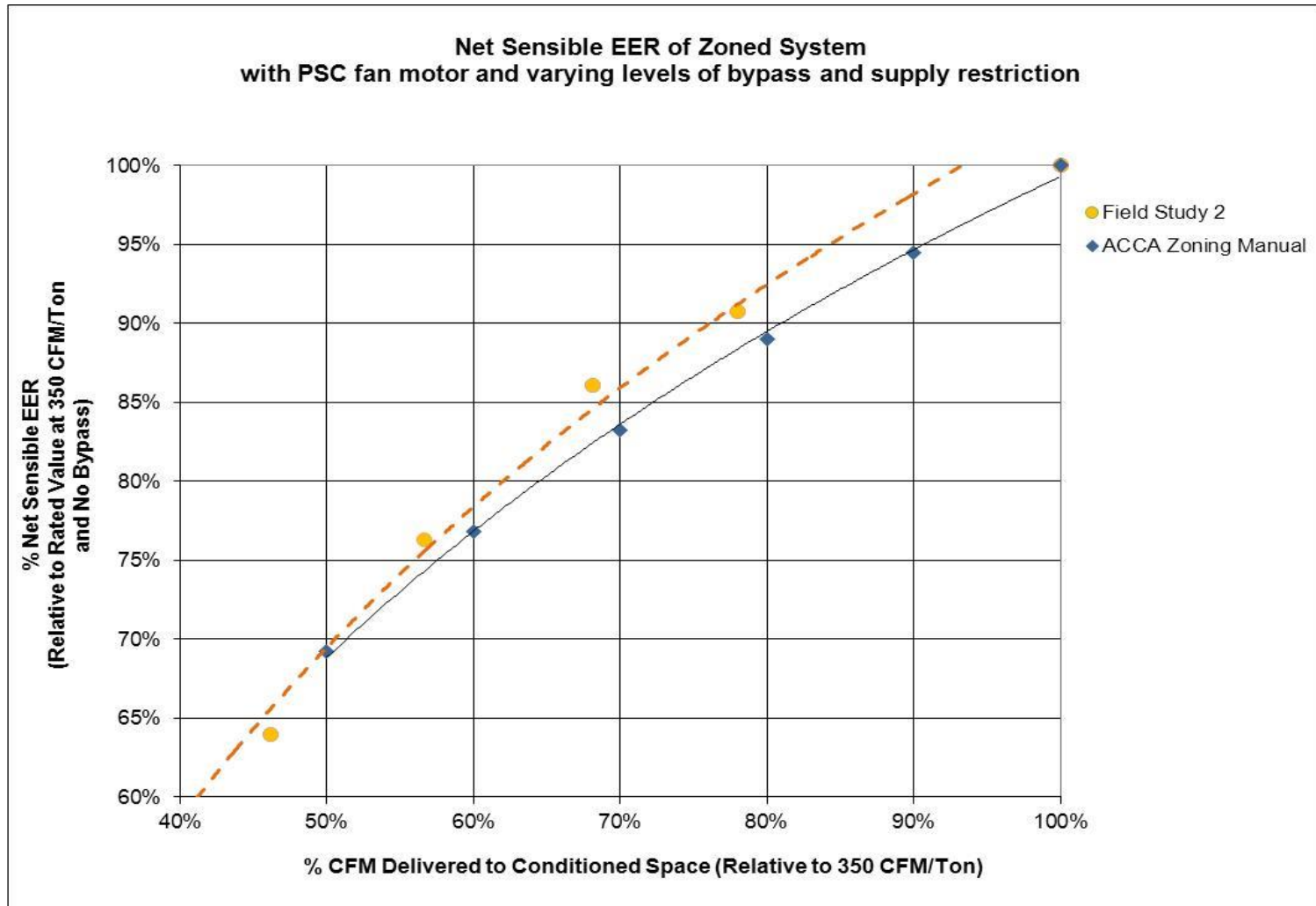
Bypass Graph

(Carrier Laboratory Tests 1993)



Bypass and Supply

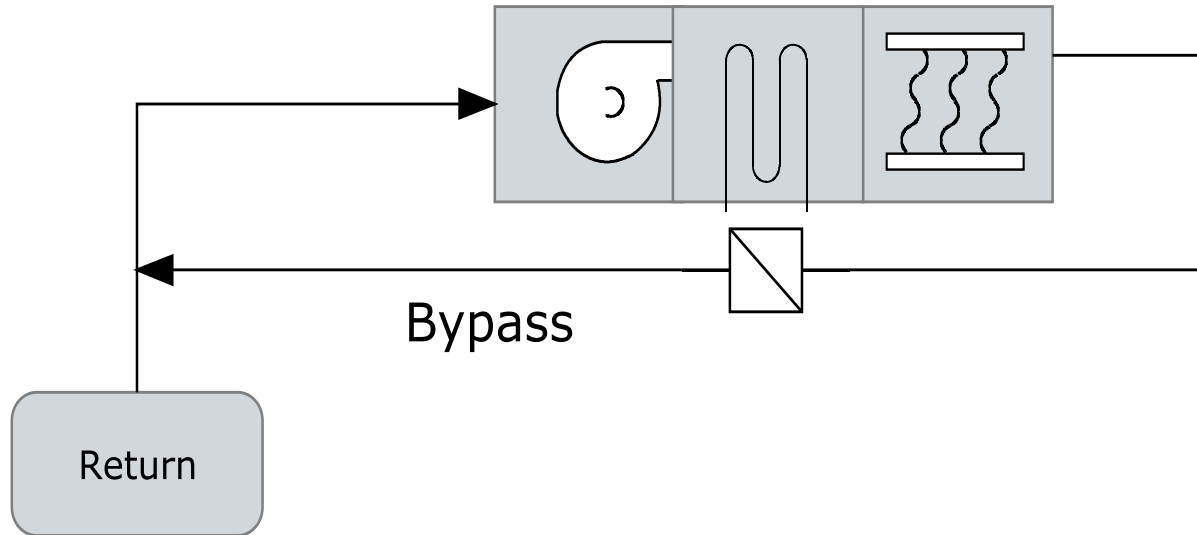
(ACCA Manual* and Field Data)



*Manual Zr, Zoned Comfort Systems for Residential Low-Rise Buildings First Edition, Version 1.00 Review Draft 8.10anc — April 21, 2011

Bypasses: Why are They so Bad?

Imagine this



Think of the equipment entering temperature
in heating and
in cooling.

Heating hot air is hard
Cooling cold air is hard

Demonstration

The image shows a man in a light blue polo shirt pointing at a whiteboard. The whiteboard contains a table with HVAC data and a diagram of a duct system. The table has the following data:

TEST						
Zone #1	0	0	0	0	C	C
Zone #2	0	0	C	C	0	0
Bypass	0	C	0	C	0	C
Flow ΔP	4.5	10.1	2.7	12.4	1.87	
Flow CFM	100	103	75	134	157	
Return T	71	72	72			

Handwritten notes on the whiteboard include "20" and "30" with arrows pointing to specific cells in the table. To the right of the man is a diagram of a duct system with handwritten notes: "0.10 - 6.62" and "0.10".

VIDEO:
ELIMINATING ZONING BYPASS SAVES 22% - 32%

Zoning with Ducts – What Works?

1. Keep it Simple
2. Use Less Ducts
3. Use Shorter Ducts (then they may not have to be larger)
4. Don't use Bypasses
5. Increase Duct Insulation – Duct Conduction is your enemy

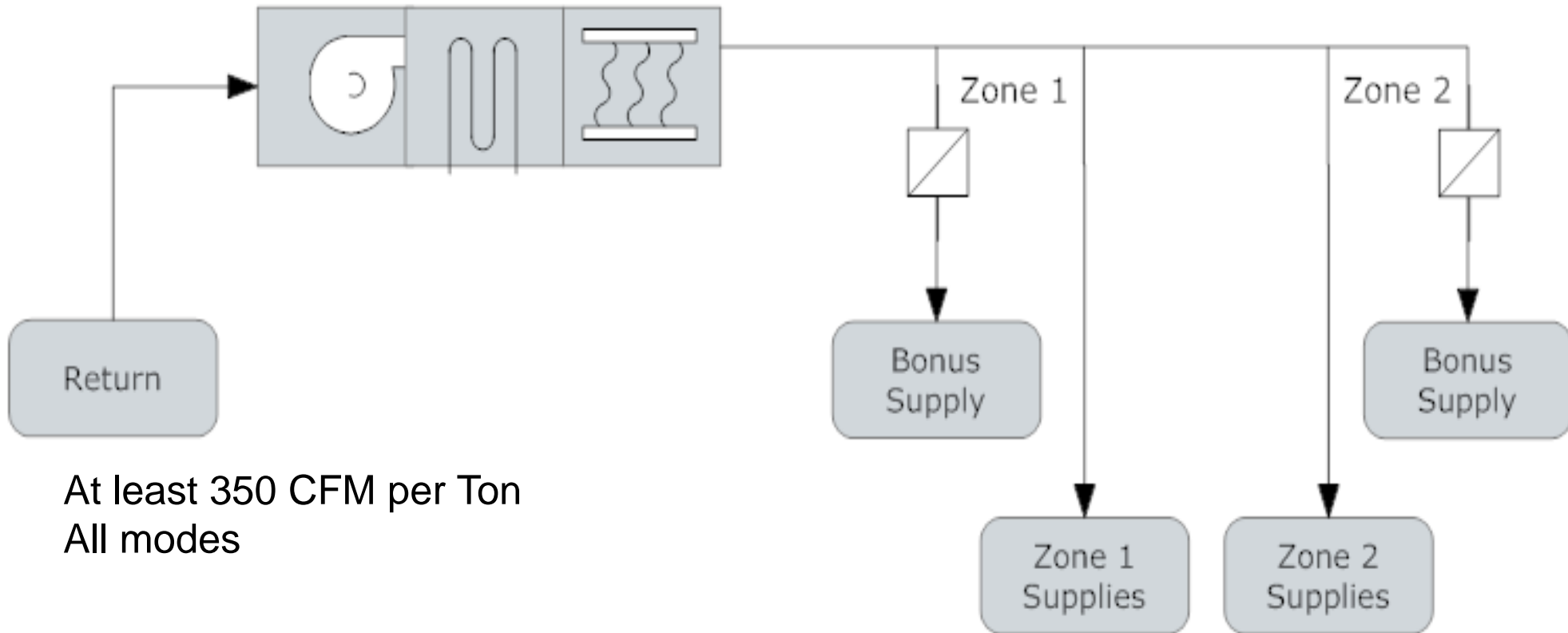
The Modern Way

How to Design a Single Capacity Multi-Zone
System that Provides Comfort and Efficiency



Single Capacity System

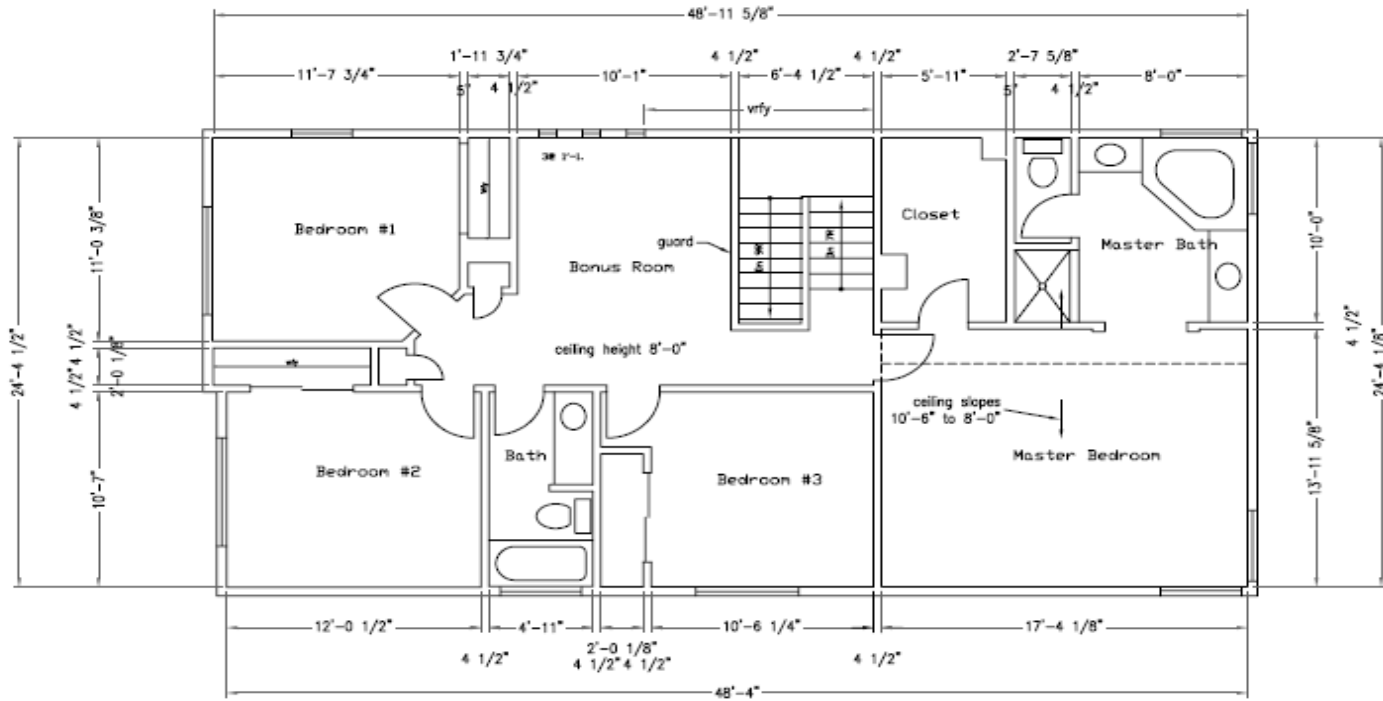
- Move Part of the Capacity from One Zone to Another based on Load



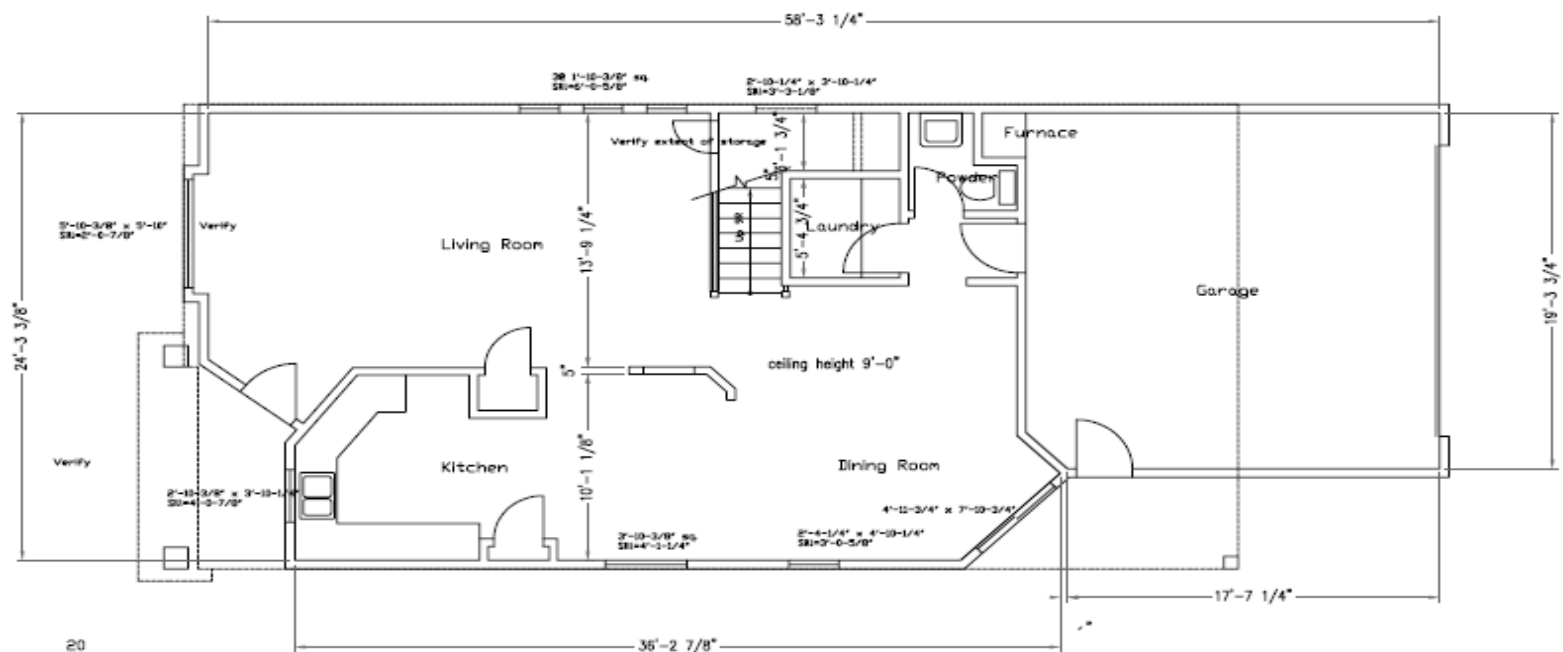
At least 350 CFM per Ton
All modes

Proper Design & Selection – Estimate Loads

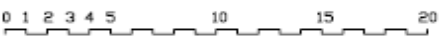
1. Estimate Design Heating Loads
2. Estimate Design Cooling Loads – Sensible and Latent
3. Determine the airflow required to each room



2 Caleb Existing Upper Floor Plan
SCALE: 1/4" = 1'-0"



1 Caleb Existing Lower Floor Plan
SCALE: 1/4" = 1'-0"



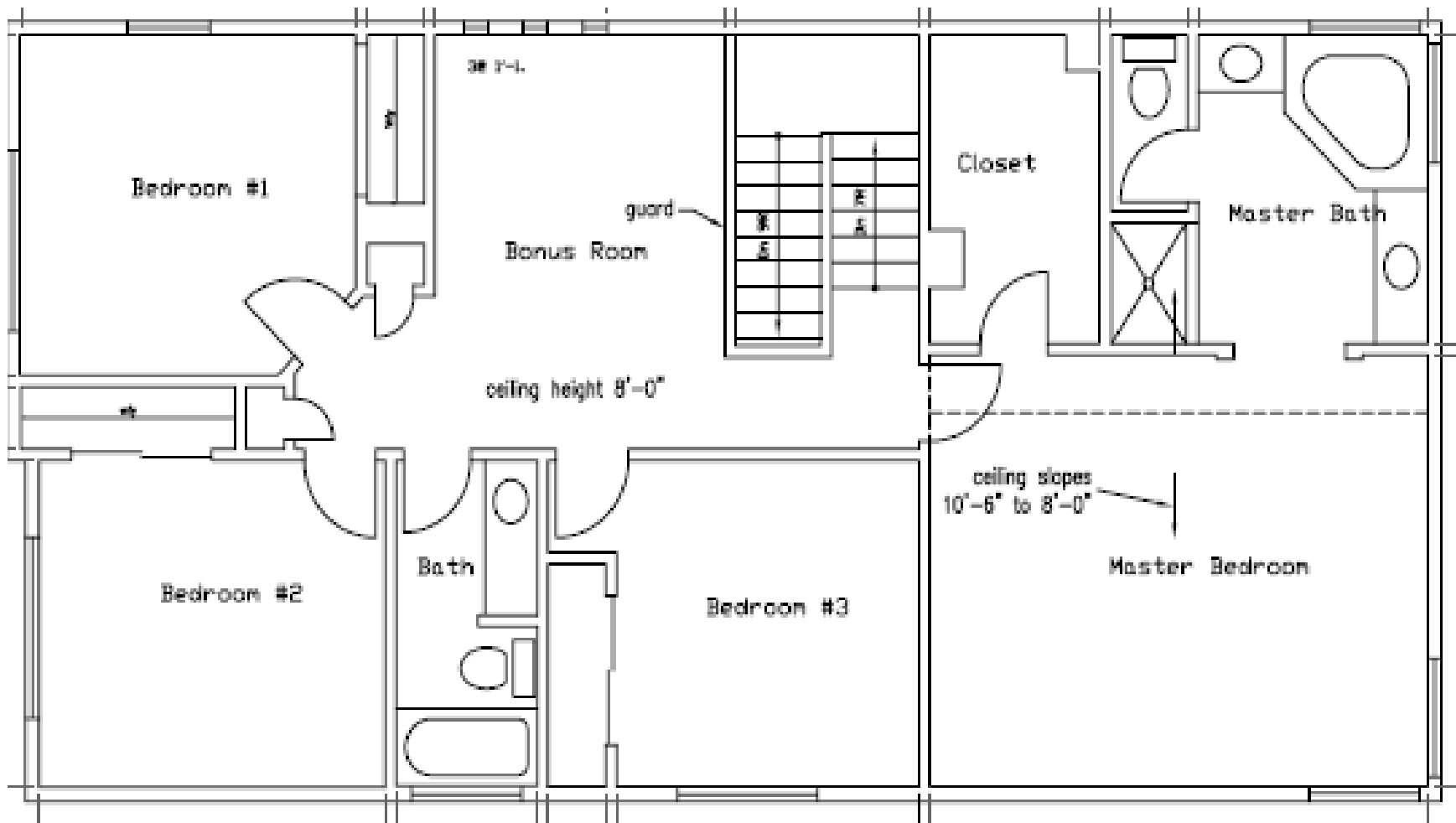
Manual J Heating and Cooling Sensible Loads

Upstairs	Cooling	Heating
Bedroom 1	1,842	1,750
Bedroom 2	1,601	1,795
Bedroom 3	1,354	990
Master BR	3,892	4,238
Common/Bth	1,434	1,324
Upstairs	10,123	10,097
Downstairs		
Living Room	3,587	6,726
Dining/Kitchen	2,878	5,706
Downstairs	6,465	12,432

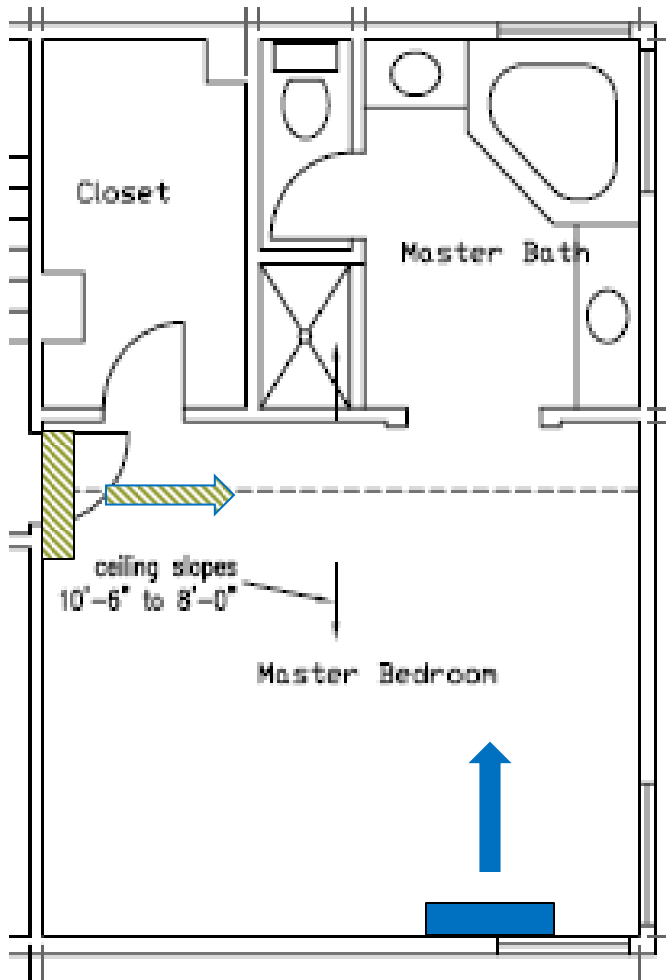
Options

- Single Zone
- Two Zones
 - Ducted Multi Split (single outside unit, three heads [could be 3 zones])
 - Ductless Mini or Multi Splits
 - Ducted Dampered Two Speed System
 - Ducted Dampered Single Speed System

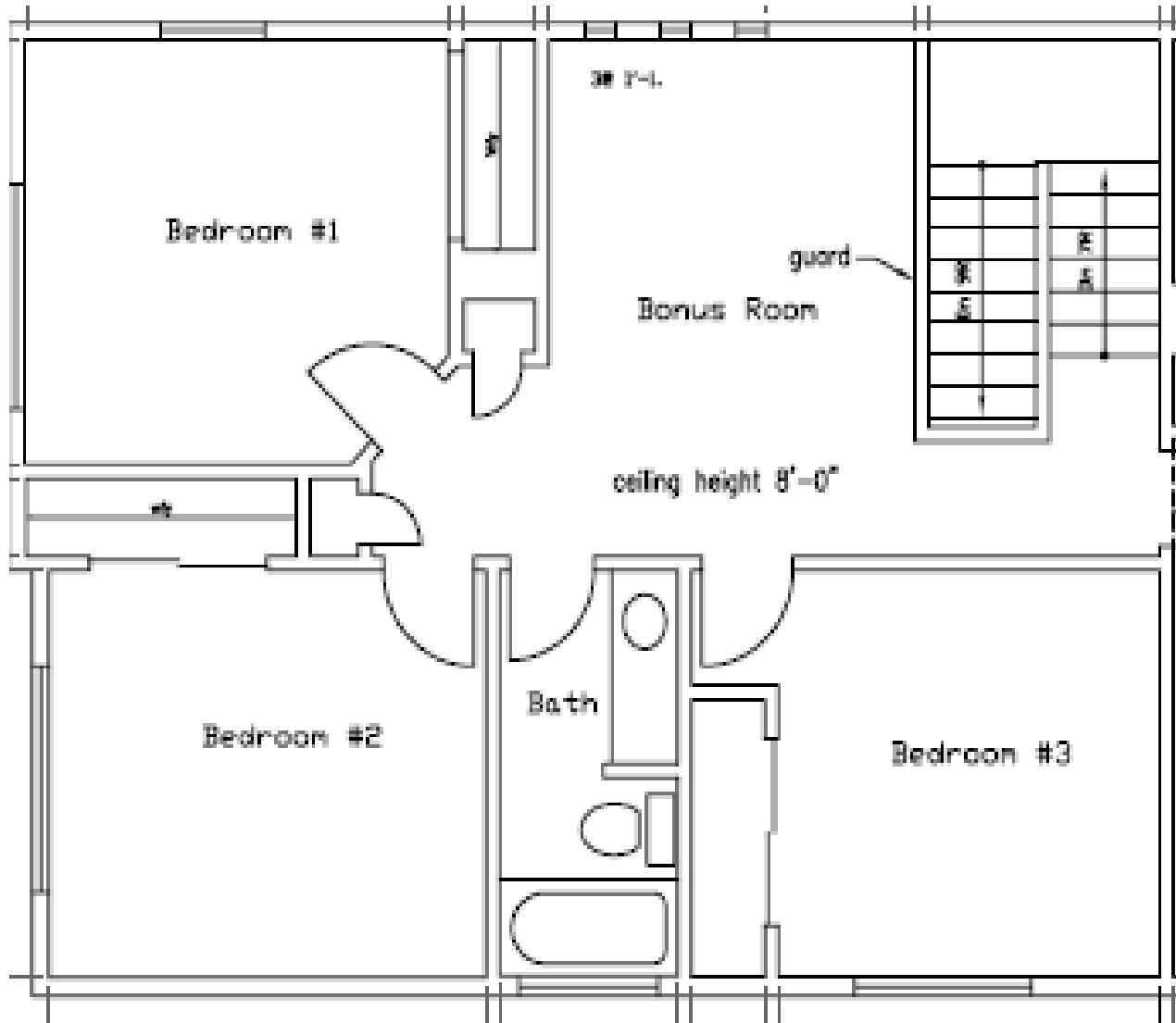
Ductless or Ducted M-split

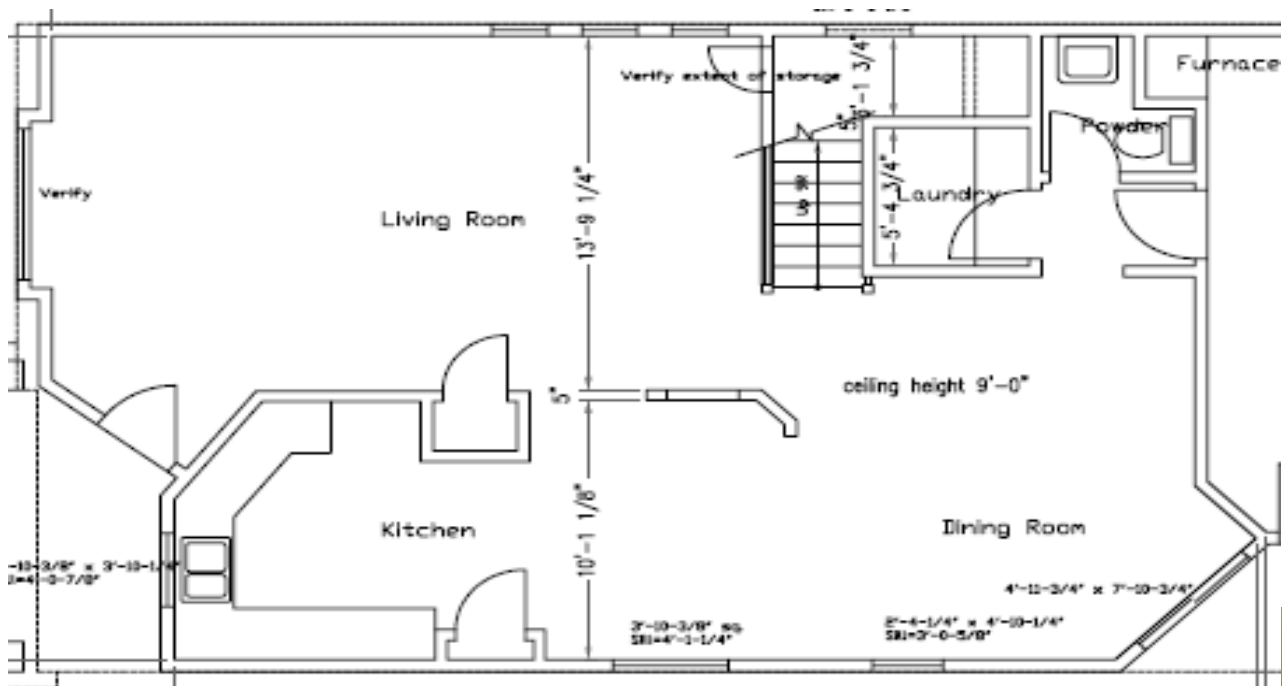


Ductless or Ducted M-split

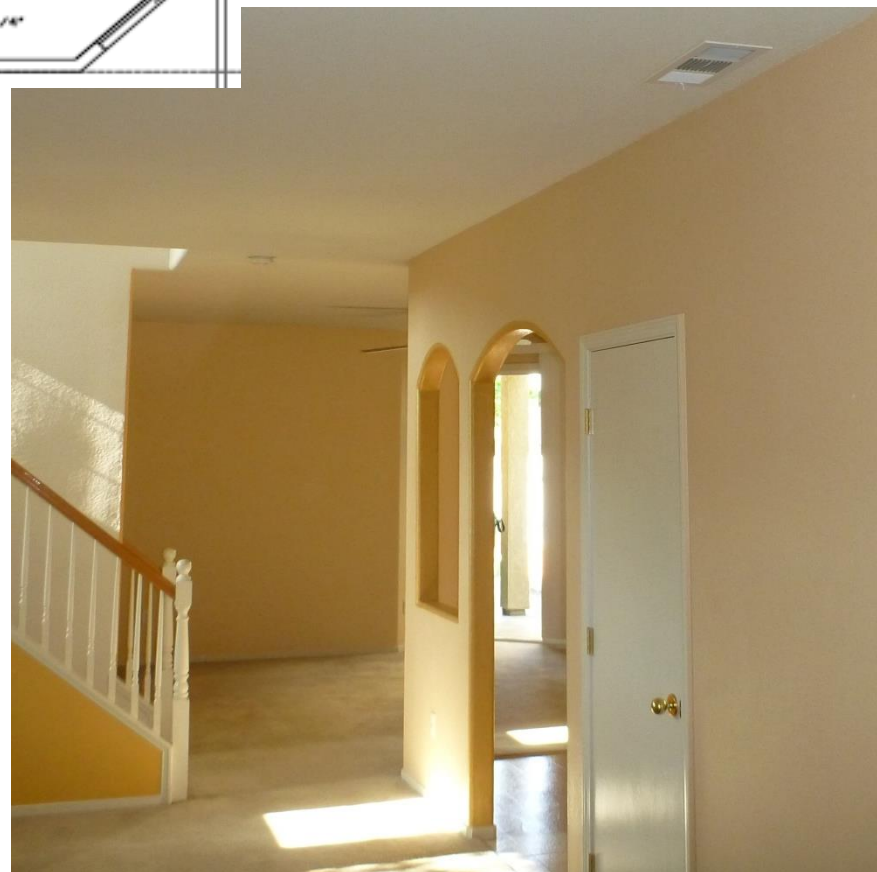


Ductless or Ducted M-split





Ductless or Ducted M-split



Options

- **Single Zone**
- Two Zones
 - Ducted Multi Split (single outside unit, three heads)
 - Ductless Mini or Multi Split
 - **Ducted Dampered Two Speed System**
 - **Ducted Dampered Single Speed System**

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Bedroom 1	1,842	1,750
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Downstairs	6,465	12,432

Calculate CFM from BTUh

Upstairs	Cooling BTUh	Heating BTUh
Total	16,588	22,529
Equipment	2 Tons	40,000

Cooling at 500 CFM per Ton = 1000 CFM
1000 CFM/16,588 BTUh = **0.060 CFM/BTUh**

Heating at 400 CFM per Ton = 800 CFM
800 CFM/22,529 BTUh = **0.036 CFM/BTUh**

Cooling CFM by Room

Upstairs	Cooling		Cool CFM
Bedroom 1	1,842	$1,842 \times 0.06$	= 111 CFM
Bedroom 2	1,601	$1,601 \times 0.06$	= 97 CFM
Bedroom 3	1,354	$1,354 \times 0.06$	= 81 CFM
Master BR	3,892	$3,892 \times 0.06$	= 234 CFM
Common/Bth	1,434	$1,433 \times 0.06$	= 86 CFM
Upstairs	10,123		
Downstairs			
Living Room	3,587	$3,587 \times 0.06$	= 215 CFM
Dining/Kitchen	2,878	$2,878 \times 0.06$	= 173 CFM
Downstairs	6,465		

Heating CFM by Room

Upstairs	Heating		Heat CFM
Bedroom 1	1,750	$1,750 \times .036$	= 62 CFM
Bedroom 2	1,795	$1,795 \times .036$	= 64 CFM
Bedroom 3	990	$990 \times .036$	= 35 CFM
Master BR	4,238	$4,238 \times .036$	= 150 CFM
Common/Bth	1,324	$1,324 \times .036$	= 47 CFM
Upstairs	10,097		
Downstairs			
Living Room	6,726	$6,726 \times .036$	= 239 CFM
Dining/Kitchen	5,706	$5,706 \times .036$	= 203 CFM
Downstairs	12,432		

Manual J Standard CFM for Single Zone

Upstairs	Cool CFM	Heat CFM	Design Single Zone CFM
Bedroom 1	111 CFM	62 CFM	111 CFM
Bedroom 2	97 CFM	64 CFM	97 CFM
Bedroom 3	81 CFM	35 CFM	81 CFM
Master BR	234 CFM	150 CFM	234 CFM
Common/Bth	86 CFM	47 CFM	86 CFM
Upstairs	610 CFM	359 CFM	610 CFM
Downstairs			
Living Room	215 CFM	239 CFM	239 CFM
Dining/Kitchen	173 CFM	203 CFM	203 CFM
Downstairs	390 CFM	441 CFM	441 CFM

Manual J Standard CFM for Single Zone

Don't like this heat cool CFM difference because of different velocities at delivery terminals

Upstairs	Cool CFM	Heat CFM	D
Bedroom 1	111 CFM	62 CFM	111 CFM
Bedroom 2	97 CFM	64 CFM	97 CFM
Bedroom 3	81 CFM	35 CFM	81 CFM
Master BR	234 CFM	150 CFM	234 CFM
Common/Bth	86 CFM	47 CFM	86 CFM
Upstairs	610 CFM	359 CFM	610 CFM
Downstairs			
Living Room	215 CFM	239 CFM	239 CFM
Dining/Kitchen	173 CFM	203 CFM	203 CFM
Downstairs	390 CFM	441 CFM	441 CFM

Re-Calculate CFM

Upstairs	Cooling BTUh	Heating BTUh
Total	16,588	22,529
Equipment	2 Tons	40,000

Cooling at 500 CFM per Ton = 1000 CFM
1000 CFM/16,588 BTUh = 0.060 CFM/BTUh

Heating at 500 CFM per Ton = 1000 CFM
1000 CFM/22,529 BTUh = 0.044 CFM/BTUh

New Heating CFM by Room

Upstairs	Heating		Heat CFM
Bedroom 1	1,750	$1,750 \times .044$	= 77 CFM
Bedroom 2	1,795	$1,795 \times .044$	= 79 CFM
Bedroom 3	990	$990 \times .044$	= 44 CFM
Master BR	4,238	$4,238 \times .044$	= 186 CFM
Common/Bth	1,324	$1,324 \times .044$	= 58 CFM
Upstairs	10,097		
Downstairs			
Living Room	6,726	$6,726 \times .044$	= 296 CFM
Dining/Kitchen	5,706	$5,706 \times .044$	= 251 CFM
Downstairs	12,432		

Manual J Standard CFM for Single Zone

Upstairs	Cool CFM	Heat CFM	Design Single Zone CFM
Bedroom 1	111 CFM	77 CFM	111 CFM
Bedroom 2	97 CFM	79 CFM	97 CFM
Bedroom 3	81 CFM	44 CFM	81 CFM
Master BR	234 CFM	186 CFM	234 CFM
Common/Bth	86 CFM	58 CFM	86 CFM
Upstairs	610 CFM	444 CFM	610 CFM
Downstairs			
Living Room	215 CFM	296 CFM	296 CFM
Dining/Kitchen	173 CFM	251 CFM	251 CFM
Downstairs	388 CFM	547 CFM	547 CFM

For a Single Zone System

Proceed with Manual D type process to:

1. Lay out terminals and duct system
2. Select terminals based on CFM, throw and room configuration
3. Calculate total effective length (TEL)
4. Calculate available static pressure (ASP) for selected equipment (change equipment selection if necessary)
5. Determine friction factor *IWC drop per 100 ft.*
($ASP / TEL \times 100$)
6. Select duct sizes based on CFM and friction factor

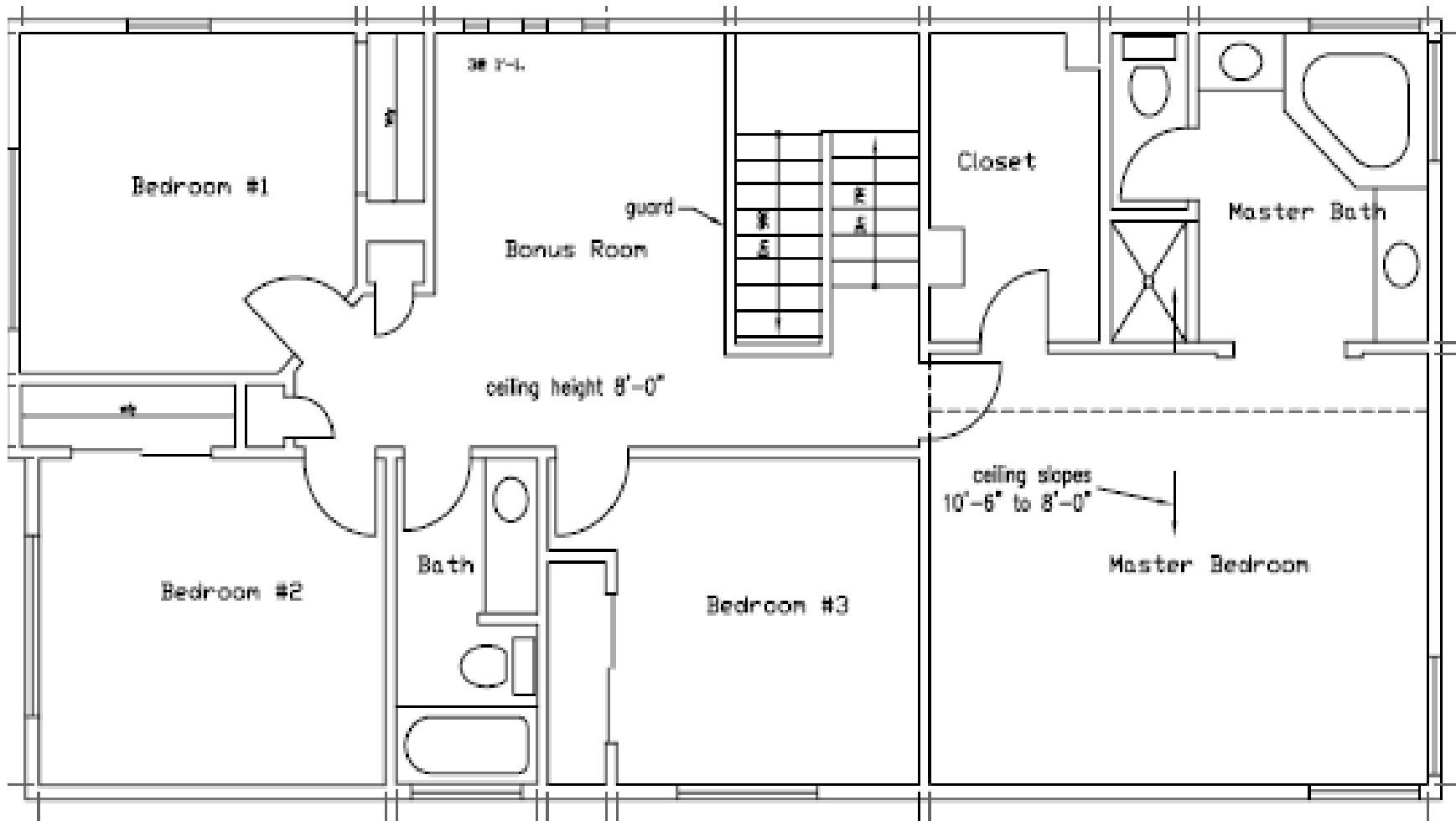
Back to the Zoning for Comfort

- Review the issue
- Review the CFM delivery desires
- Look at ducted zoning options

Manual J Heating and Cooling Sensible Loads

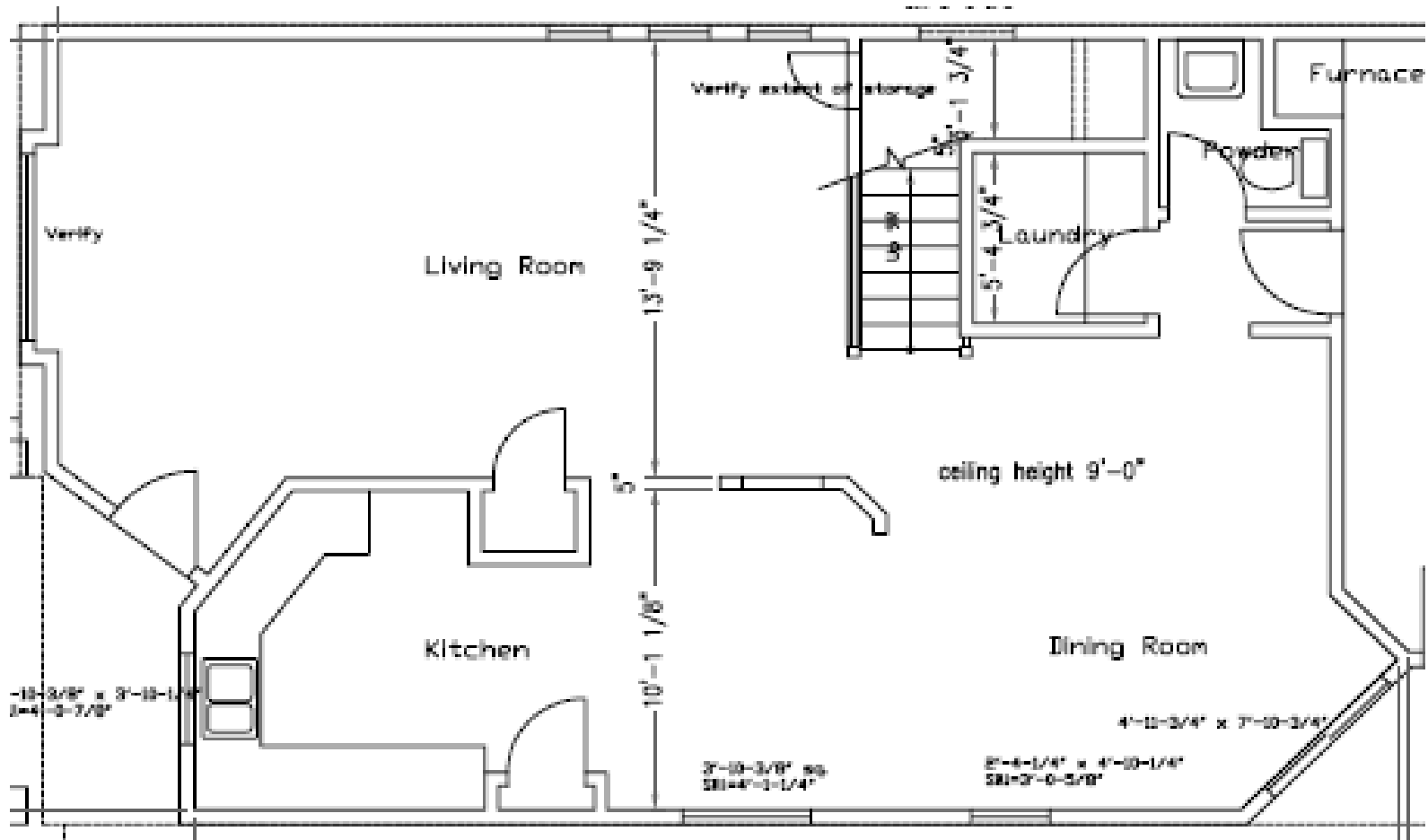
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Where Shall We Put the Upstairs Bonus Supply?



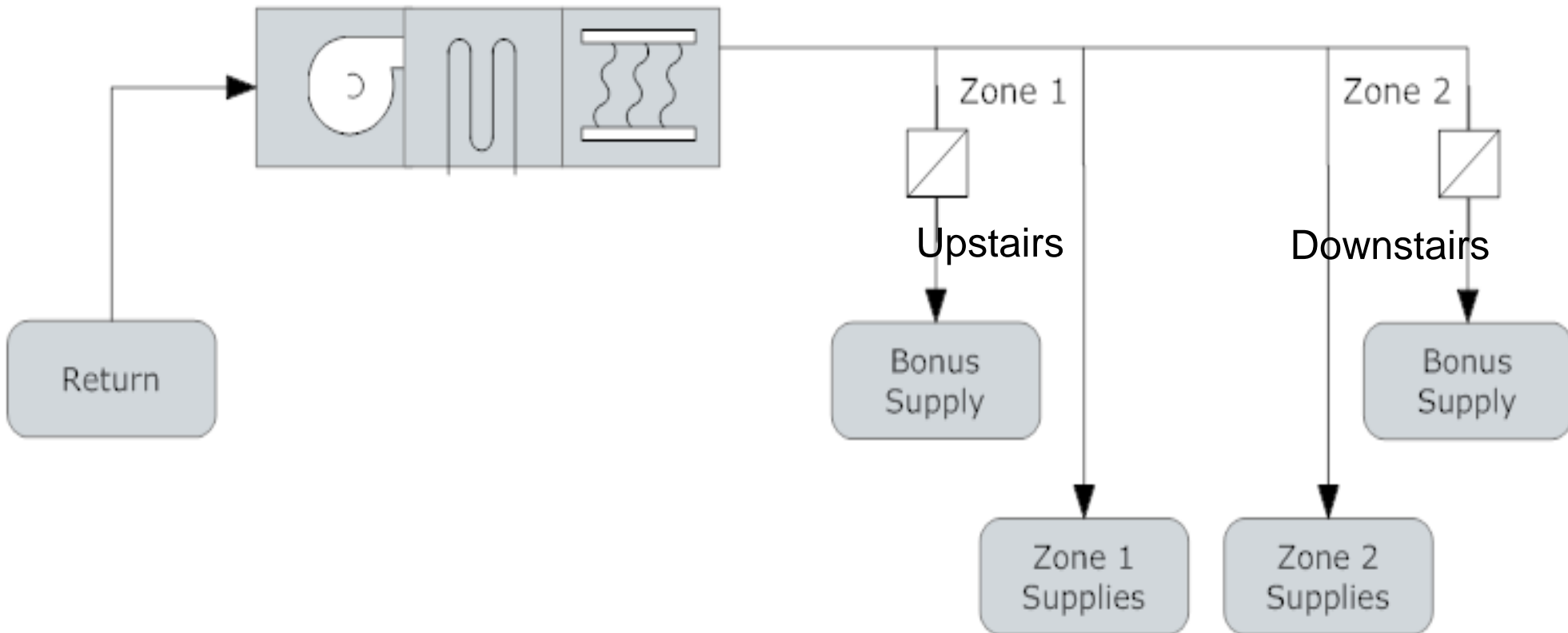
It needs to be where the thermostat is

Where Shall We Put the Downstairs Bonus Supply?



Single Capacity Zoned System

- Move Part of the Capacity from One Zone to Another based on Load (Capacity Shift Zoning)



Manual J Standard CFM

(at 500 CFM per ton cooling and heating)

Upstairs	Cool CFM	Heat CFM	Design Single Zone CFM
Bedroom 1	111 CFM	77 CFM	111 CFM
Bedroom 2	97 CFM	79 CFM	97 CFM
Bedroom 3	81 CFM	44 CFM	81 CFM
Master BR	234 CFM	186 CFM	234 CFM
Common/Bth	86 CFM	58 CFM	86 CFM
Upstairs	610 CFM	444 CFM	610 CFM
Downstairs			
Living Room	215 CFM	296 CFM	296 CFM
Dining/Kitchen	173 CFM	251 CFM	251 CFM
Downstairs	388 CFM	547 CFM	547 CFM

Shift Capacity

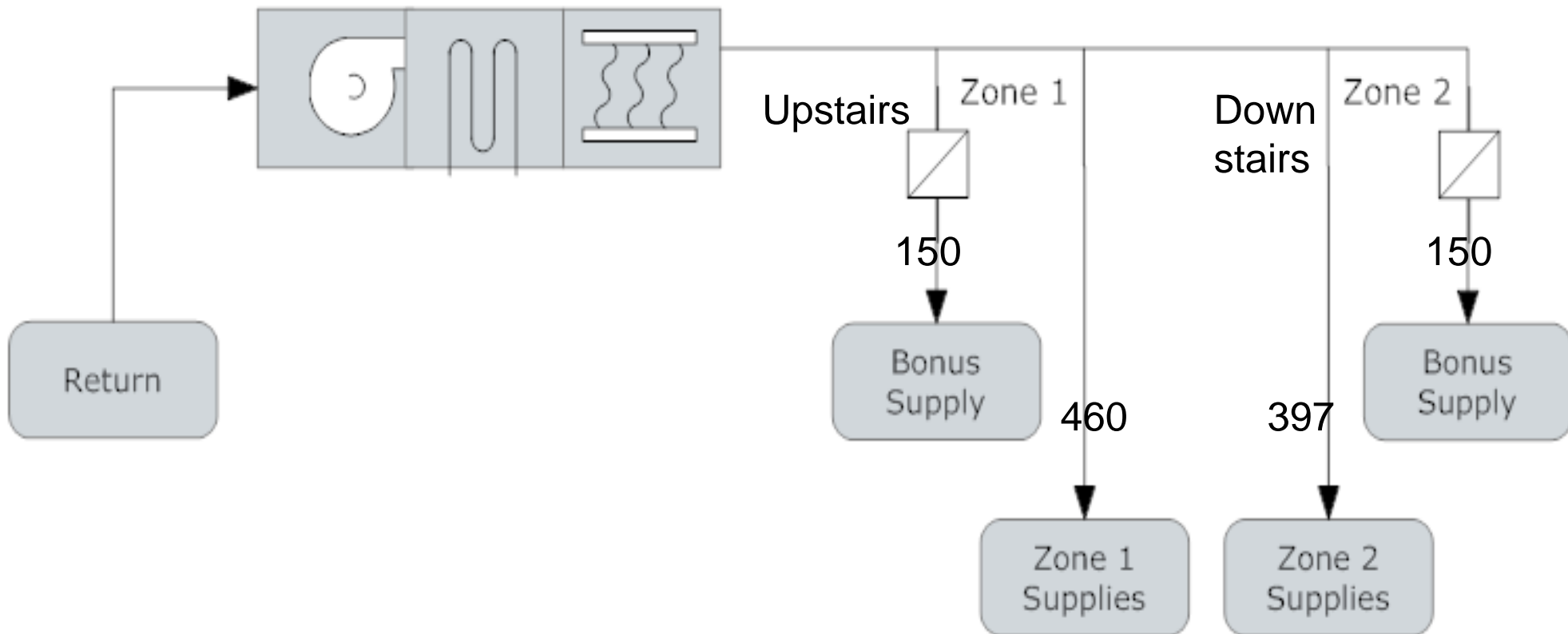
- Recommended Shift 20% to 30%
- In this case 20% of 610 CFM = 120 CFM
- 30% of 547 CFM = 164 CFM
- Call it 150 Shifted CFM to Bonus Supplies

Determine CFM to Always Open Supplies:

- Upstairs Supplies $610 - 150 = 460$ CFM
- Downstairs Supplies $547 - 150 = 397$ CFM

Single Capacity Zoned System

- Move Part of the Capacity from One Zone to Another based on Load



Re-estimate Supply Flows

- We moved 150 CFM out of the always open supplies, so they don't have to be as big
- So what is the new upstairs flow (% of old)?

$$\underline{\hspace{2cm}} / \underline{\hspace{2cm}} = \underline{\hspace{2cm}}$$

- So what is the new downstairs flow (% of old)?

$$\underline{\hspace{2cm}} / \underline{\hspace{2cm}} = \underline{\hspace{2cm}}$$

Upstairs	“Std” CFM	Zone CFM
Bedroom 1	111	× .754 = 84
Bedroom 2	97	× .754 = 73
Bedroom 3	81	× .754 = 61
Master BR	234	× .754 = 176
Common/Bth	86	× .754 = 65
Master BR Bonus		75 CFM
Common/Bath Bonus		75 CFM
Upstairs	610	609
Downstairs		
Living Room	296	× .725 = 215
Dining/Kitchen	251	× .725 = 182
Living Bonus		150 CFM
Downstairs	547	547

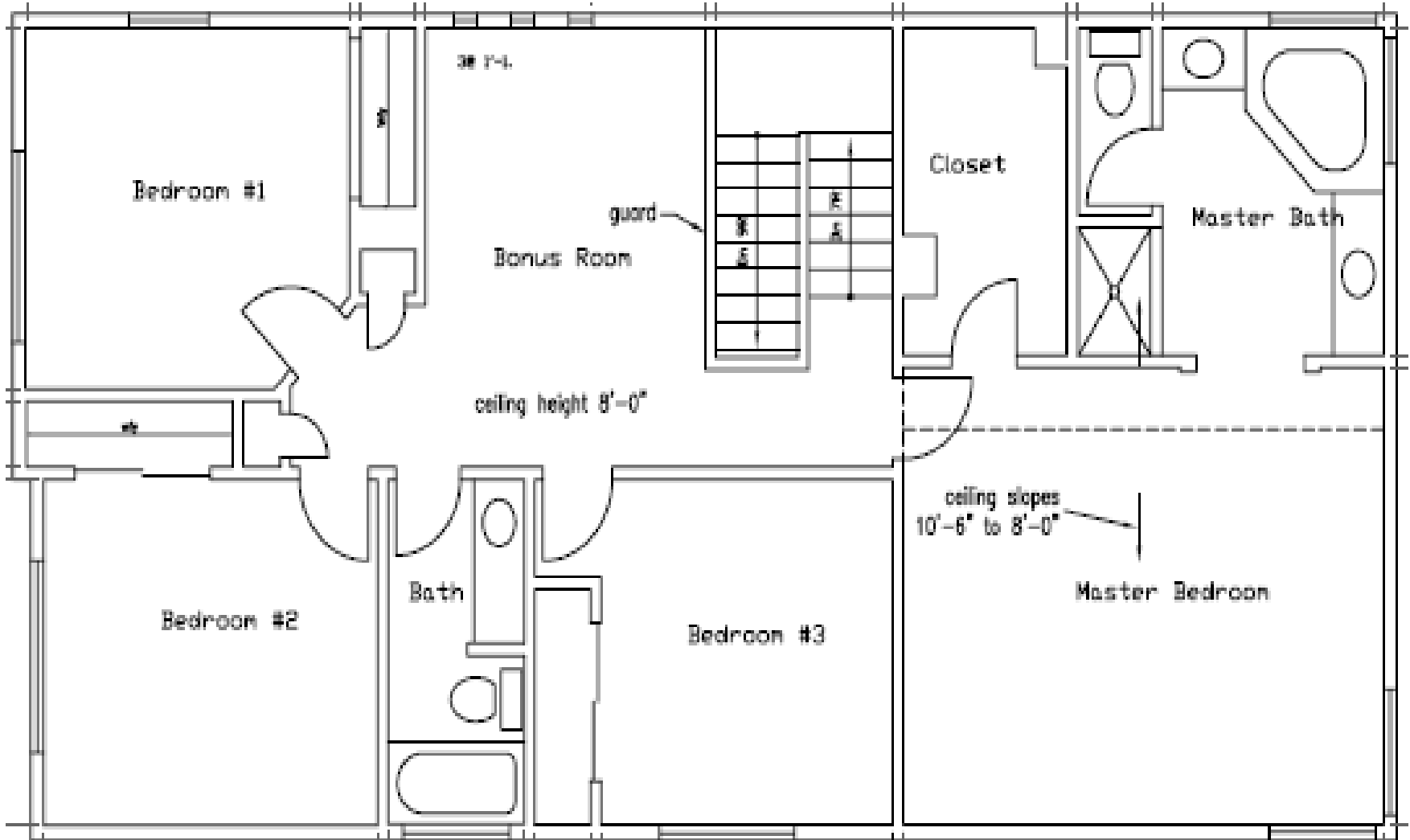
Proper Design

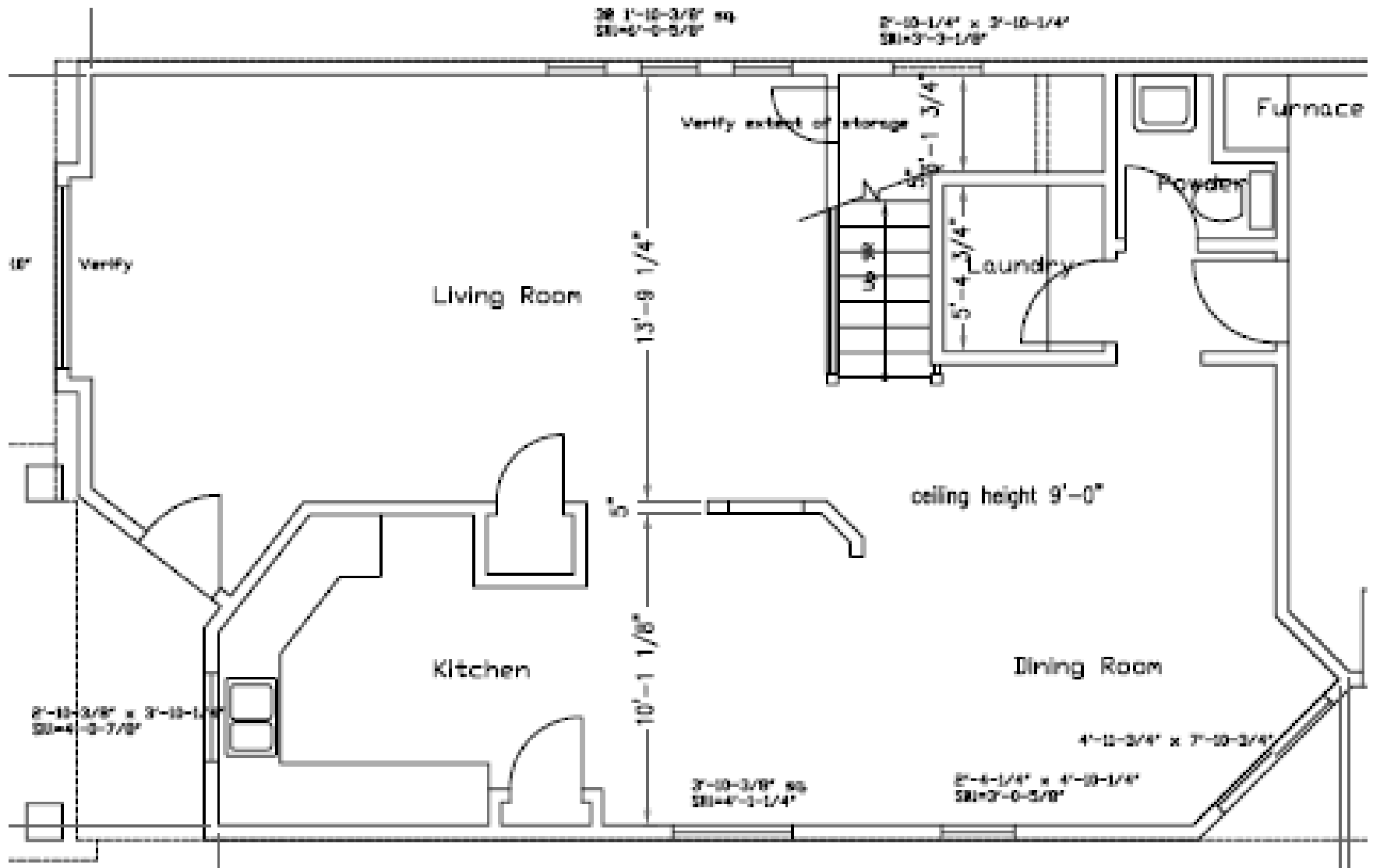
1. Determine the best locations and size for the air terminals (registers)

Use ACCA Manual T

2. Design the duct system to the available pressure and minimize the effective length

Use ACCA Manual D or Equivalent





To Size Our Ducts We Need to Know How
Much Static Pressure We Have

To do that we need to know the air handler and
coil we will use

We start with the Load and CFM

	Cooling BTUh	Heating BTUh
Upstairs	10,123	10,097
Downstairs	6,465	12,432
Total	16,588	22,529
CFM	1000	1000

Select the AC Unit

DETAILED COOLING CAPACITIES# CONTINUED

EVAPORATOR AIR		CONDENSER ENTERING AIR T		
		95 (35)		
CFM	EWB °F (°C)	Capacity MBtuh		Total System KW**
		Total	Sens†	
1) 31 Outdoor Section With CAP**241				
700	72 (22.2)	24.63	12.41	2.06
	67 (19.4)	22.71	15.56	2.05
	63 (17.2)††	21.29	15.11	2.05
	62 (16.7)	20.99	18.67	2.05
	57 (13.9)	20.79	20.79	2.05
	72 (22.2)	25.10	13.63	2.13
900	67 (19.4)	23.21	17.59	2.13
	63 (17.2)††	21.82	17.01	2.12
	62 (16.7)	22.11	22.11	2.12
	57 (13.9)	22.11	22.11	2.12

COOLING INDOOR MODEL	CAPACITY	POWER
*CAP**2414A**	1.00	1.00
CAP**2417A**	1.00	1.00
CAP**3014A**	1.01	1.01
CAP**3017A**	1.01	1.01
CAP**3617A**	1.01	1.01
CAP**3619A**	1.01	1.01
CAP**3621A**	1.02	1.02
CNPH*2417A**	1.00	1.00
CNPH*3017A**	1.01	1.01
CNPH*3117A**	1.02	1.02
CNPV*2414A**	1.00	1.00
CNPV*2417A**	1.00	1.00
CNPV*3014A**	1.01	1.01
CNPV*3017A**	1.01	1.01
CNPV*3117A**	1.02	1.02
CSPH*2412A**	1.00	1.00
CSPH*3012A**	1.01	1.01
FB4CNF024	0.99	0.99

What matching Coils are there?

Select the AC Coil

Looking for low Static Pressure Drop

COIL STATIC PRESSURE DROP (in. w.c.) PURO

UNIT SIZE	S								
	400	500	600	700	800	900	1000	1100	1200
2417	0.048	0.068	0.090	0.112	0.140	0.170	0.203		
	0.064	0.091	0.122	0.150	0.188	0.224	0.263		
3017	0.042	0.060	0.080	0.102	0.128	0.157	0.188	0.222	0.255
	0.055	0.076	0.104	0.127	0.158	0.190	0.225	0.266	0.306
3117	0.031	0.046	0.063	0.083	0.105	0.130	0.156	0.193	0.229
	0.039	0.056	0.075	0.097	0.121	0.149	0.179	0.212	0.246
3617	0.043	0.061	0.082	0.103	0.128	0.157	0.189	0.221	0.255
	0.056	0.079	0.107	0.133	0.166	0.200	0.236	0.276	0.316

CNPHP / CNRHP

PERFORMANCE

COIL STATIC PRESSURE DROP (in. w.c.) PURON®

CAPMP / CARMP

UNIT SIZE	STANDARD								
	400	500	600	700	800	900	1000	1100	1200
2417	0.056	0.076	0.097	0.123	0.151	0.182	0.215		
	0.060	0.082	0.105	0.132	0.163	0.195	0.231		
3014	0.054	0.077	0.102	0.133	0.167	0.206	0.248	0.296	0.347
	0.059	0.084	0.111	0.142	0.181	0.223	0.269	0.319	0.375
3017	0.043	0.059	0.077	0.096	0.119	0.144	0.171	0.201	0.232
	0.046	0.063	0.083	0.105	0.130	0.157	0.186	0.219	0.252
3614	0.047	0.069	0.093	0.119	0.151	0.187	0.227	0.270	0.317
	0.053	0.076	0.101	0.129	0.162	0.200	0.241	0.286	0.335
3617	0.023	0.036	0.052	0.069	0.089	0.110	0.135	0.160	0.189
	0.042	0.058	0.076	0.095	0.117	0.142	0.169	0.198	0.231
3621	0.026	0.037	0.050	0.062	0.077	0.092	0.109	0.128	0.148
	0.029	0.040	0.053	0.065	0.082	0.099	0.119	0.138	0.160

Select the Furnace for CFM and Watt Draw (BPM preferred)

AIR DELIVERY - CFM (With Filter)

UNIT SIZE	RETURN-AIR SUPPLY	SPEED	EXTERNAL STATIC PRESSURE (In. W.C.)									
			0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9	1.0
045-08	Bottom or Side(s)	High	1085	1035	975	915	845	770	675	565	390	195
		Med-High	920	875	830	770	710	640	555	440	250	—
		Med-Low	820	775	730	680	620	555	470	360	190	—
045-12	Bottom or Side(s) Low	High	1450	1375	1305	1225	1145	1050	955	845	705	510
		Med-High	1360	1300	1240	1175	1115	1040	950	850	725	575
		Med-Low	1250	1210	1160	1100	1040	965	885	790	670	520
070-08	Bottom or Side(s)	High	1030	1010	980	945	900	845	775	680	490	335
		Med-High	835	815	790	760	720	675	610	490	375	265
		Med-Low	725	700	675	645	600	555	475	390	300	—
070-12	Bottom or Side(s)	High	1425	1375	1305	1225	1145	1050	955	845	705	510
		Med-High	1320	1260	1200	1135	1075	1000	915	815	695	520
		Med-Low	1200	1150	1100	1040	990	920	840	745	635	480
070-16	Bottom or Side(s)	High	1805	1755	1685	1605	1525	1435	1345	1255	1165	1075
		Med-High	1630	1580	1510	1430	1350	1265	1175	1085	1000	910
		Med-Low	1460	1410	1340	1260	1180	1100	1015	925	840	755
090-14	Bottom or Side(s)	High	1650	1600	1535	1465	1385	1285	1175	1055	895	645
		Med-High	1515	1485	1440	1380	1300	1220	1115	990	830	600
		Med-Low	1385	1360	1320	1260	1195	1120	1025	915	710	565
090-16	Bottom or Side(s)	High	2060	1985	1915	1820	1720	1610	1490	1340	1135	925
		Med-High	1790	1765	1715	1645	1560	1470	1345	1195	1010	820
		Med-Low	1505	1505	1480	1440	1375	1300	1190	1045	890	740
Bottom Only	High	2405	2310	2220	2130	2025	1920	1790	1660	1530	1380	
	Med-High	2225	2155	2080	1995	1895	1785	1675	1565	1420	1260	
	Med-Low	2020	1955	1880	1805	1730	1630	1535	1420	1275	1135	

This Furnace has a 14" Cabinet

58STA/STX

Final Furnace and Coil Selections

- Furnace
 - 045-12
 - Flow = 1040 CFM @ 0.5 IWC (Medium Low)
 - Replace PSC with BPM or Select a Different Furnace
- Diffuser Transition
 - 14" to 21"
- Evaporator Coil
 - 3621
 - $\Delta P = 0.12$ IWC @ 1000 CFM

Determine the Available Static Pressure for the Supply Side

- Total at Air Handler 0.50
 - Return -0.17
 - A Coil -0.12
 - Terminal -0.02

AVAILABLE 0.19

This is available for supply trunks and standard branches using Manual D methods.

- Control Damper -0.05

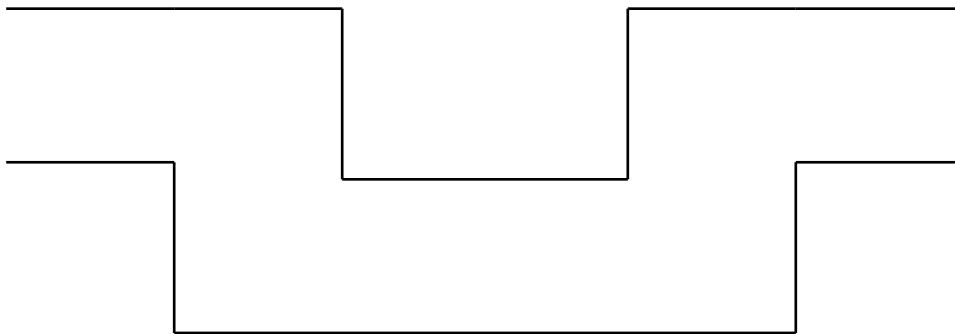
Available for Bonus Runs 0.14

How Long is the Duct System? (from an air molecule's view)

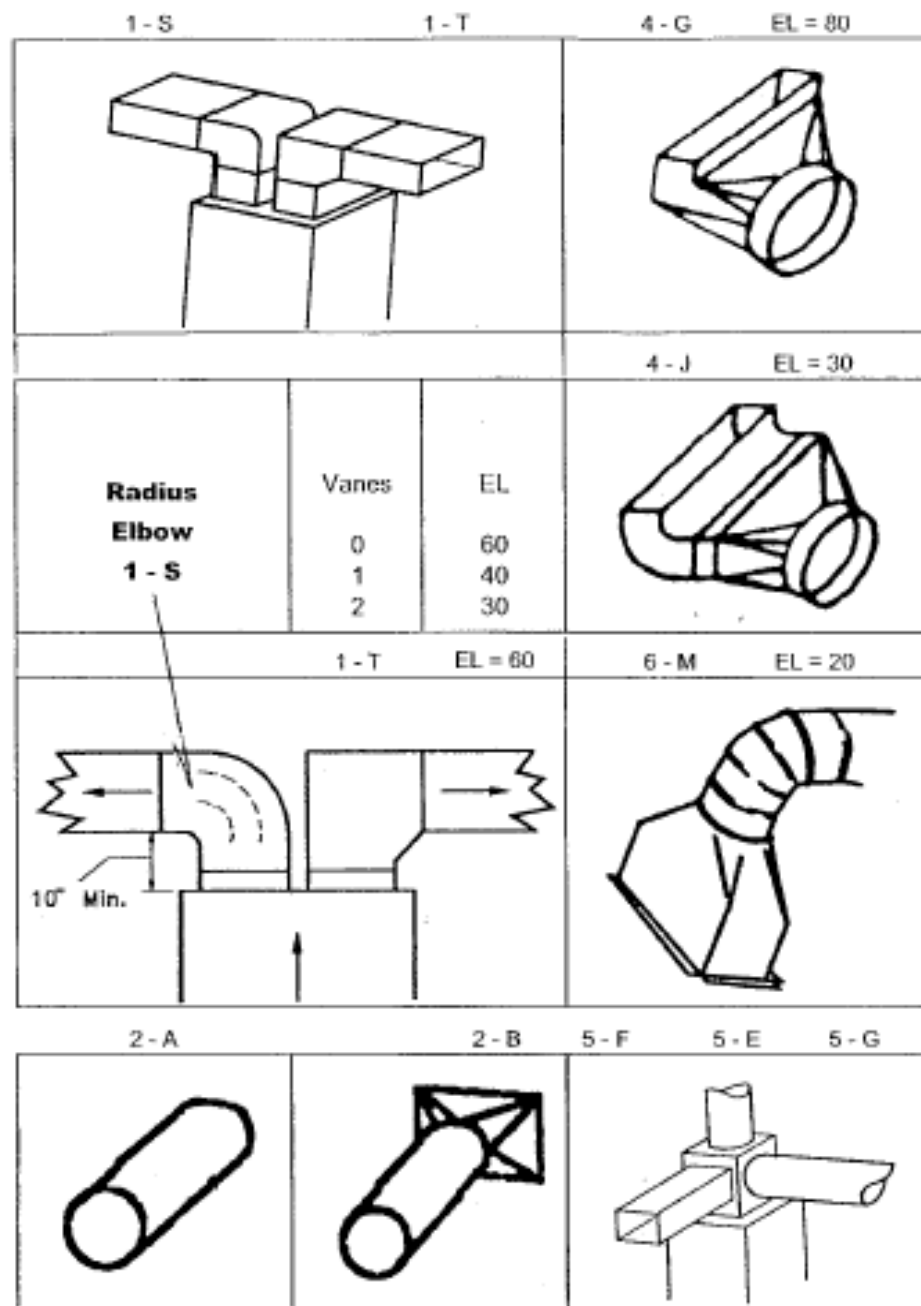
- A 200 ft. straight pipe is 200 feet long



- This 4 ft. section of pipe is how long?



330 Feet



*Excerpted from ACCA Manual D

Determine the Total Effective Length (TEL)

- Zone 1 Always Open

Straight Duct	30
Fittings	100
<hr/>	
	130
- Zone 1 Bonus 120
- Zone 2 Always Open 100
- **Zone 2 Bonus 150**

Calculate the Friction Rate

Zone 1 and Zone 2 except Bonus Runs

Available Supply Static Pressure

0.19 IWC

Longest Equivalent Length

130 Ft.

Friction Rate

$$\begin{aligned} \text{FR} &= 0.19 / 130 * 100 \\ &= 0.138 \text{ IWC per 100 Ft.} \end{aligned}$$

Bonus Runs

Available Supply Static Pressure

0.14 IWC

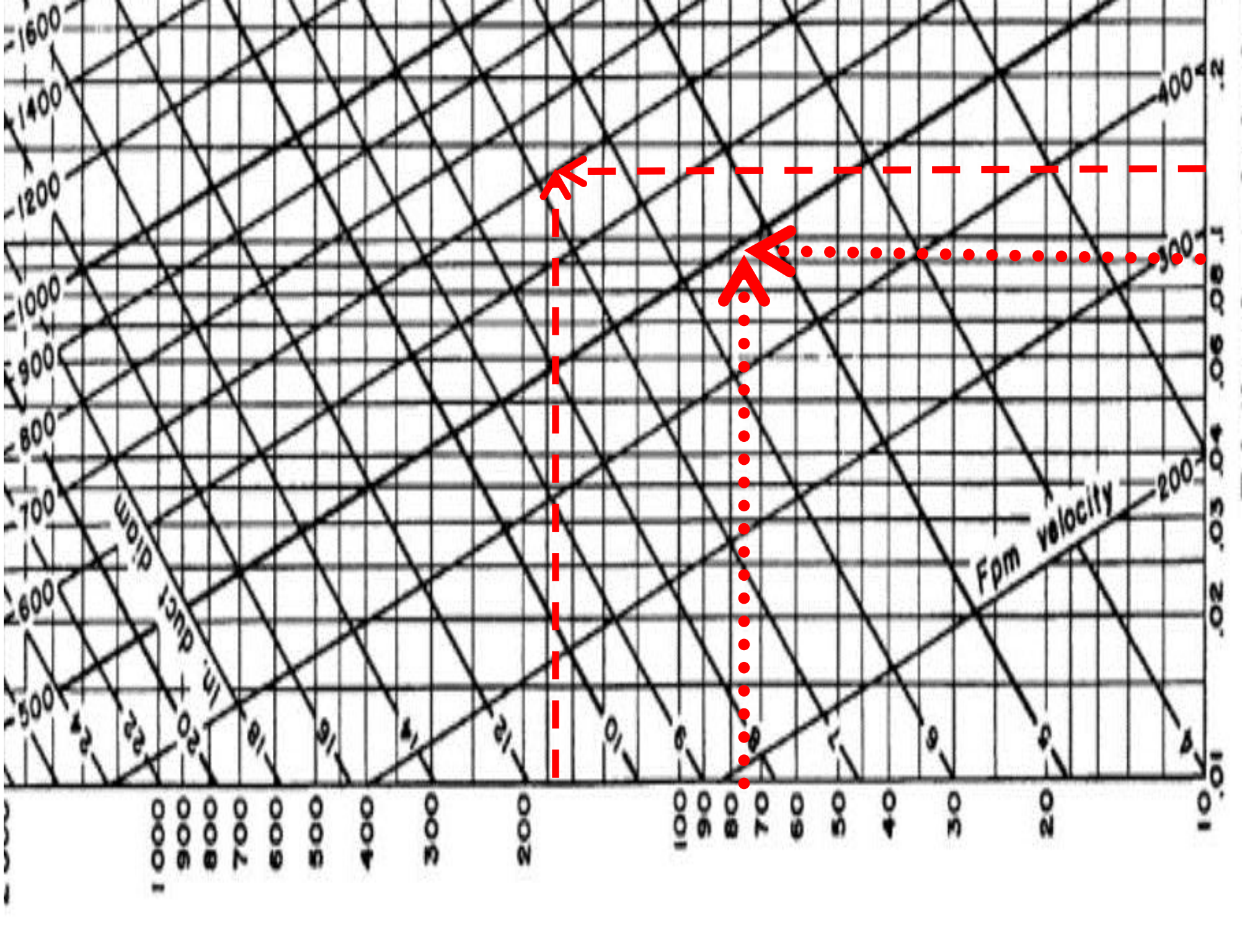
Longest Equivalent Length

150 Ft.

Friction Rate

$$\begin{aligned} \text{FR} &= 0.14 / 150 * 100 \\ &= 0.093 \text{ IWC per 100 Ft.} \end{aligned}$$

Upstairs	Zone CFM
Bedroom 1	84
Bedroom 2	73
Bedroom 3	61
Master BR	176
Common/Bth	65
Master BR Bonus	75
Common/Bath Bonus	75
Upstairs	609
Downstairs	
Living Room	215
Dining/Kitchen	182
Living Bonus	150
Downstairs	547



A Good Duct Design Provides

- Quiet
- Efficient
- Comfort

Efficient

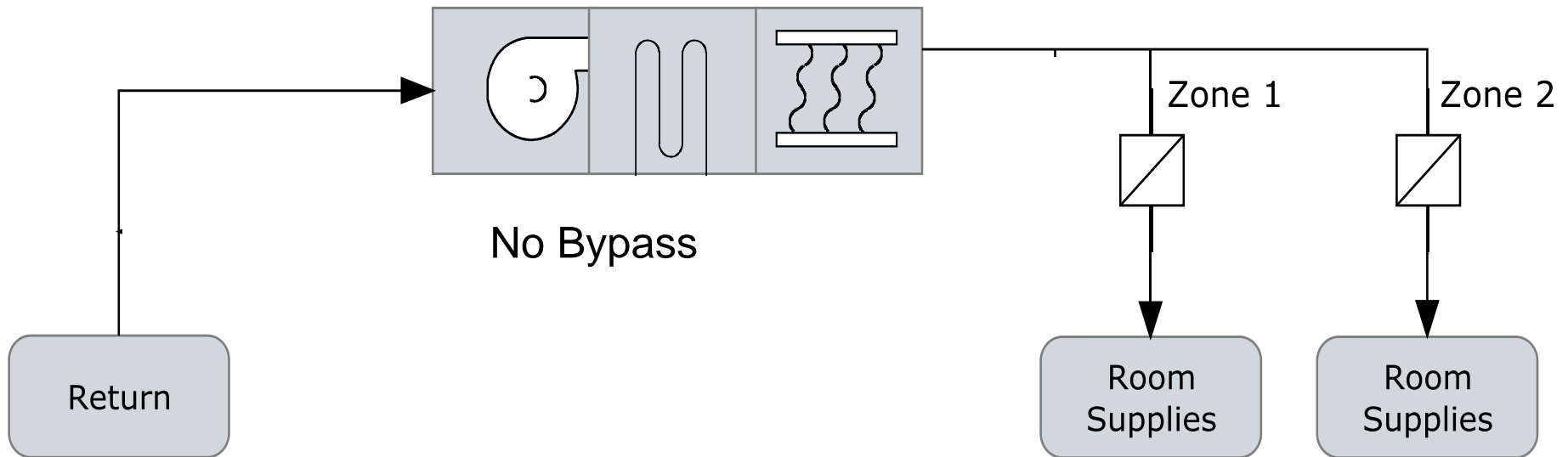
- **FAN WATT DRAW REDUCTION**
- **SUFFICIENT AIRFLOW**
 - USE METAL DUCT AND **FITTINGS**
- **LOW CONDUCTION LOSS**
 - SHORT RUNS
 - FEWER LARGER DIAMETER RUNS
 - PUT INSIDE THE CONDITIONED SPACE or INSULATE THE HECK OUT OF IT

So Does It Work?

- Test Airflow in all configurations
 - Zone 1 Only Calling
 - Zone 2 Only Calling
 - All Zones Calling
 - Every Combination
- Test at Returns
 - Flow Grid
 - Pressure Matching
 - Powered Flow Hood
 - Passive Flow Hood

A Less Desirable Solution

- Damper stop relief
- Intentionally Leaky Dampers
- Use Oval Damper Door if Possible
- Keep Dampers away from Supply Terminals



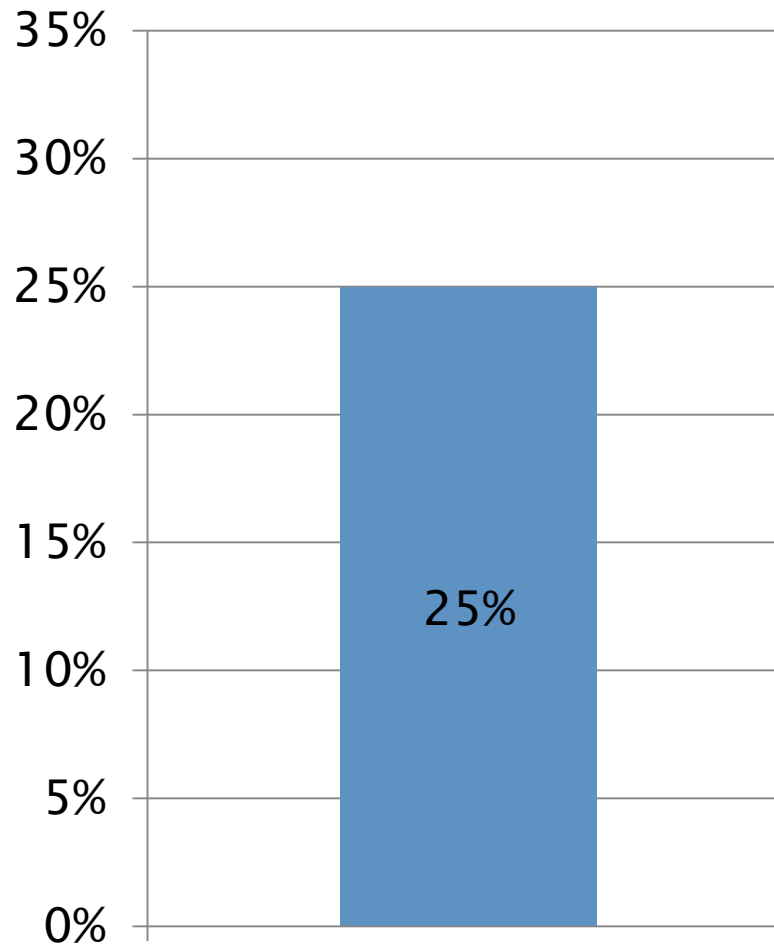
Reviewing Problems with the Old Way using Bypass

Average Energy Impact

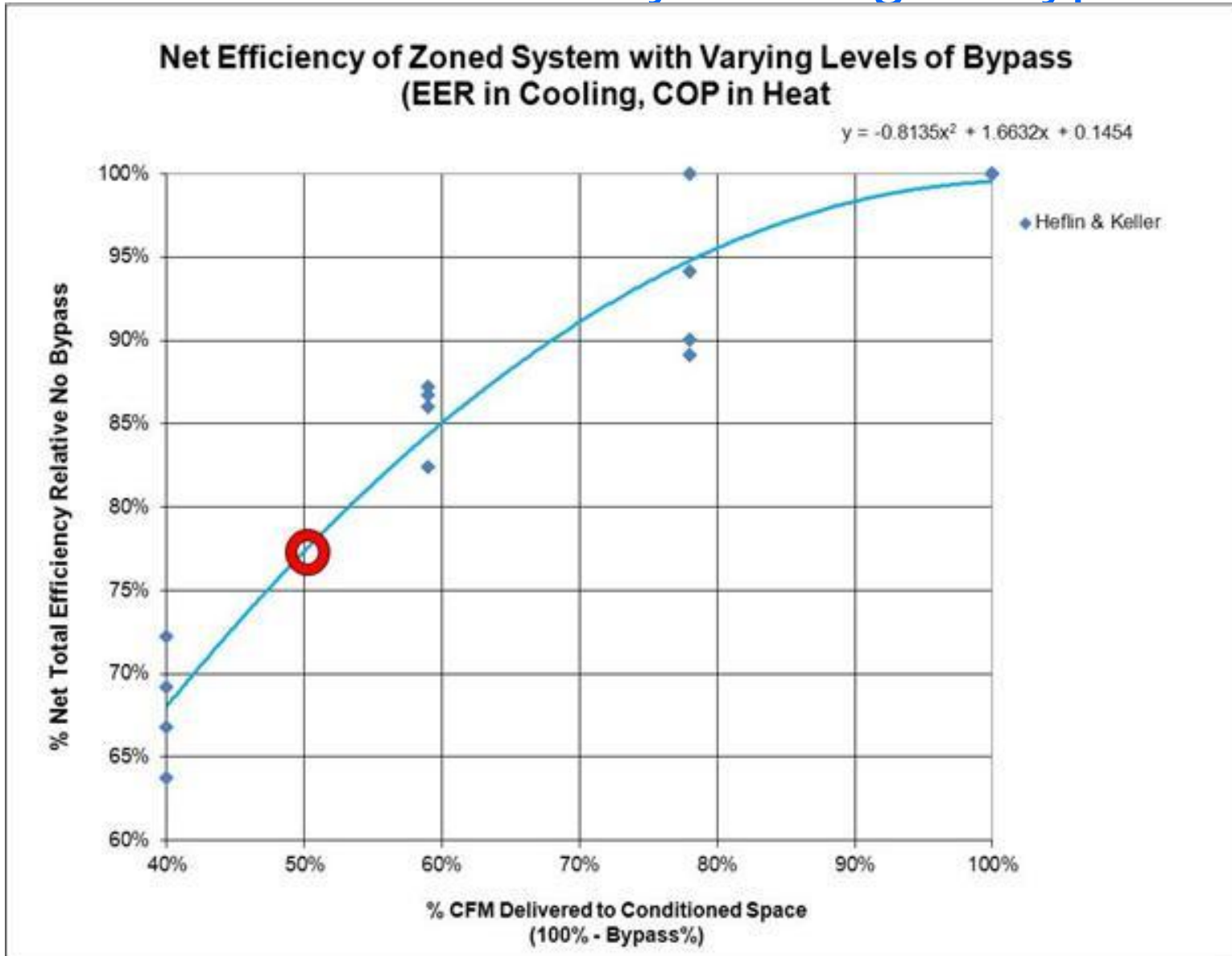
- In typical homes with dampered multi-zone systems:
 - Air Conditioning SEER and EER are degraded by 17%
 - Furnace AFUE is degraded by 4.4%

Eliminate Bypass and Obtain a Minimum Airflow

Efficiency Increase



Carrier Laboratory Testing of Bypass



Carrier ASHRAE Paper Literature Review

- Papers cited by AHRI to support the zoning performance credit
 - “All but one study employed variable or two speed equipment.”
 - “... the homes were unoccupied and zone separation (closed doors) was maintained throughout testing caused energy losses to be minimized. Thus, the documented field studies could be considered a ‘best case scenario’ in terms of energy savings.”

More Information

- <http://www.proctoreng.com/energy-efficiency/zones.html>
- <http://www.proctoreng.com/dnld/H114.pdf>
- <http://www.energy.ca.gov/2014publications/CEC-400-2014-005/CEC-400-2014-005.pdf>